

# Control Systems

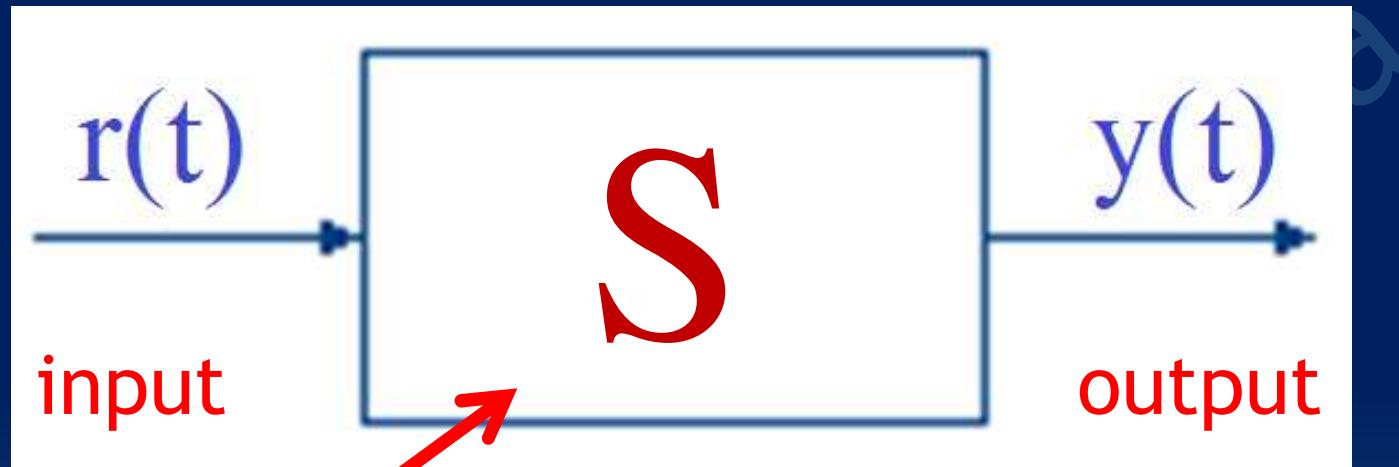
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"Time domain analysis"

part II – 2<sup>nd</sup> order systems

J. A. M. Felippe de Souza

## Time domain analysis - 2<sup>nd</sup> order systems

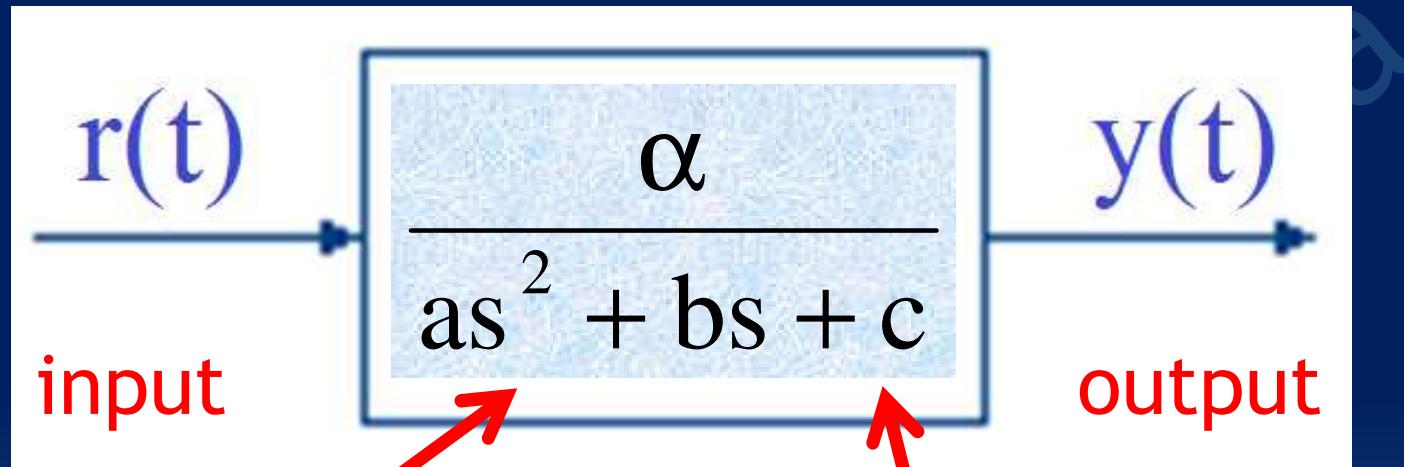


Second order systems

of the type

$$G(s) = \frac{\alpha}{as^2 + bs + c}$$

## Time domain analysis - 2<sup>nd</sup> order systems

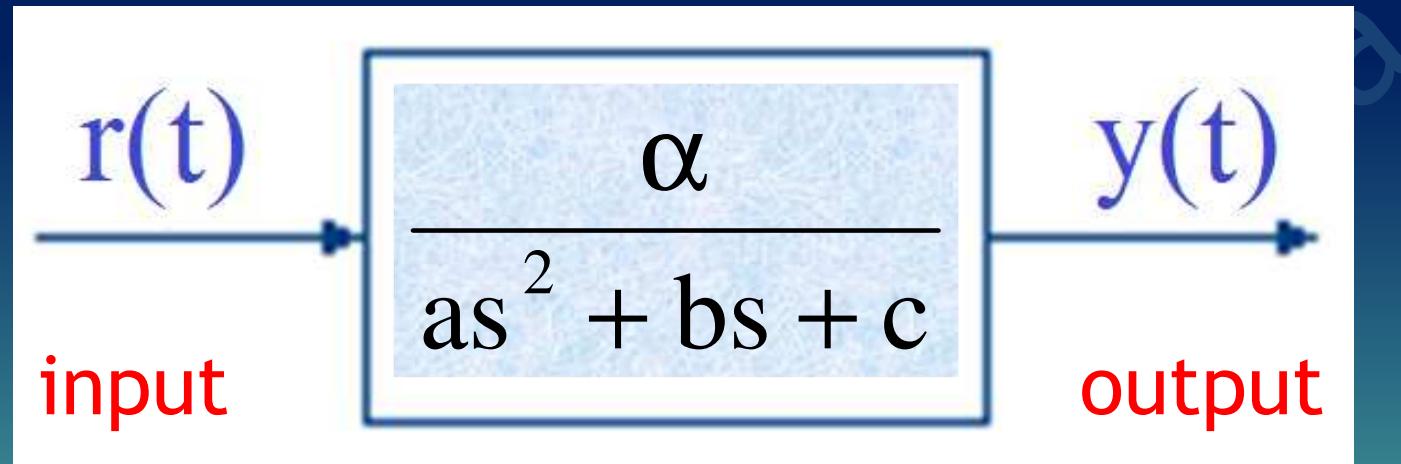


Second order systems

of the type

$$G(s) = \frac{\alpha}{as^2 + bs + c}$$

## Second order systems:



that is:

$$\frac{Y(s)}{R(s)} = \frac{\alpha}{as^2 + bs + c}$$

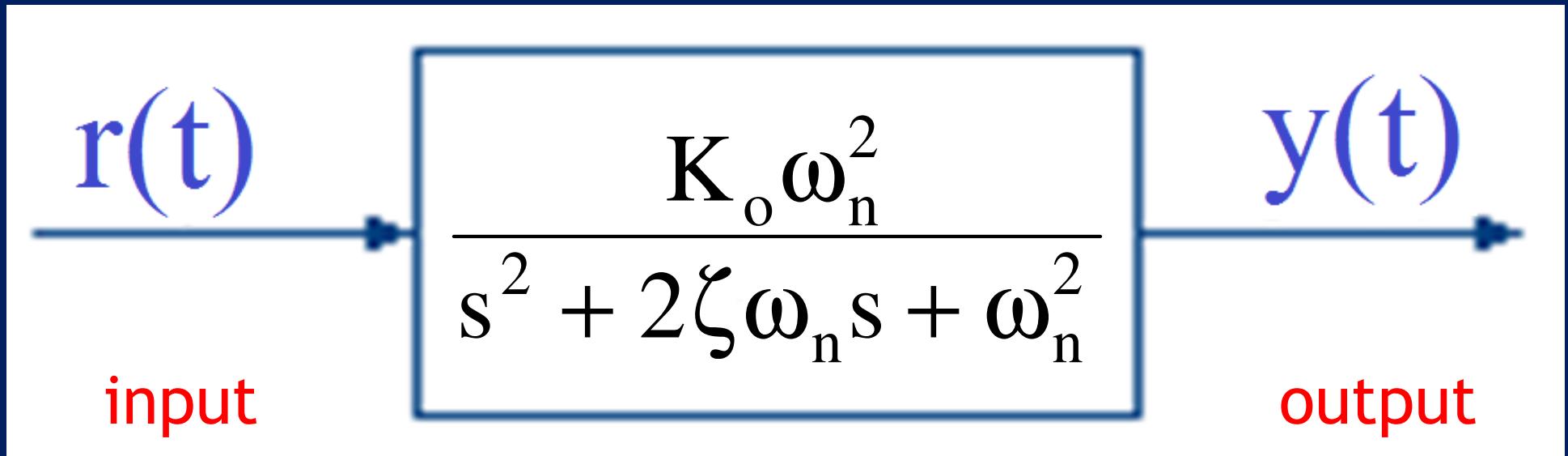
**Proof.** 

$$\frac{Y(s)}{R(s)} = \frac{\alpha}{s^2 + \frac{b}{a}s + \frac{c}{a}}$$

Annotations with red arrows and circles:

- An arrow points from the term  $\frac{\alpha}{a}$  to the coefficient  $K_0\omega_n^2$ .
- An arrow points from the term  $\frac{b}{a}$  to the coefficient  $2\zeta\omega_n$ .
- An arrow points from the term  $\frac{c}{a}$  to the coefficient  $\omega_n^2$ .

# Time domain analysis - 2<sup>nd</sup> order systems



$K_o$  = gain of the system

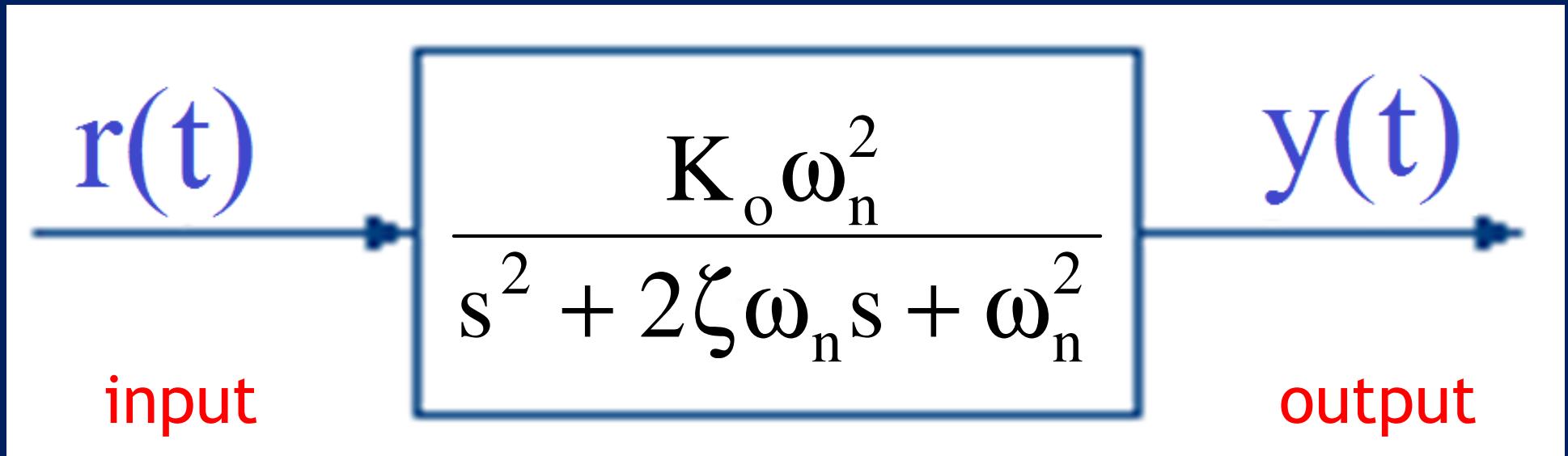
$\zeta$  = damping coefficient

$\omega_n$  = natural frequency

the *transfer function* can be rewritten as:

$$\frac{Y(s)}{R(s)} = \frac{K_o \omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

# Time domain analysis - 2<sup>nd</sup> order systems



$K_o$  = gain of the system

$\zeta$  = damping coefficient

$\omega_n$  = natural frequency

Besides these 3 above parameters, we also have

$\omega_d$  = damping frequency

$$\omega_d = \omega_n \cdot \sqrt{1 - \zeta^2} \quad 0 < \zeta \leq 1$$

### Example 1:

$$\frac{Y(s)}{R(s)} = \frac{3}{4s^2 + 12s + 1}$$

poles:  
 $s = -2,914$   
 $s = -0,086$

poles are  
real and  
distinct

$$K_o = 3 \quad \zeta = 3 \quad \omega_n = 0,5$$

### Example 2:

$$\frac{Y(s)}{R(s)} = \frac{3}{s^2 + 2s + 1}$$

poles:  
 $s = -1$   
(double)

$$K_o = 3 \quad \zeta = 1 \quad \omega_n = 1 \quad \omega_d = 0$$

poles are  
real and  
repeated

### Example 3:

$$\frac{Y(s)}{R(s)} = \frac{3}{2s^2 + 2s + 2}$$

$$K_o = 1,5 \quad \zeta = 0,5 \quad \omega_n = 1 \quad \omega_d = 0,866$$

poles:  
 $s = -0,5 \pm 0,866j$

complex  
conjugate  
poles

### Example 4:

$$\frac{Y(s)}{R(s)} = \frac{3}{s^2 + 1}$$

$$K_o = 3 \quad \zeta = 0 \quad \omega_n = \omega_d = 1$$

poles:  
 $s = \pm j$   
(pure imaginary)

complex  
conjugate  
poles

Characteristic equation:

$$p(s) = s^2 + 2\zeta\omega_n s + \omega_n^2$$

$$\begin{aligned}\Delta &= 4\zeta^2\omega_n^2 - 4\omega_n^2 = \\ &= 4\omega_n^2(\zeta^2 - 1)\end{aligned}$$

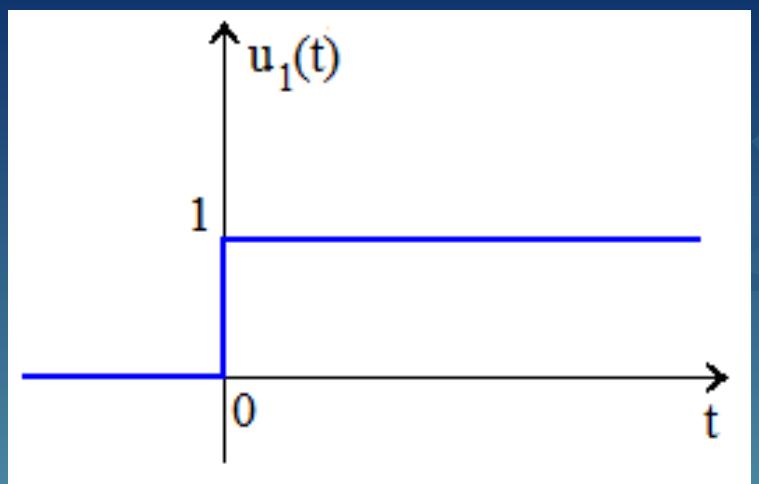
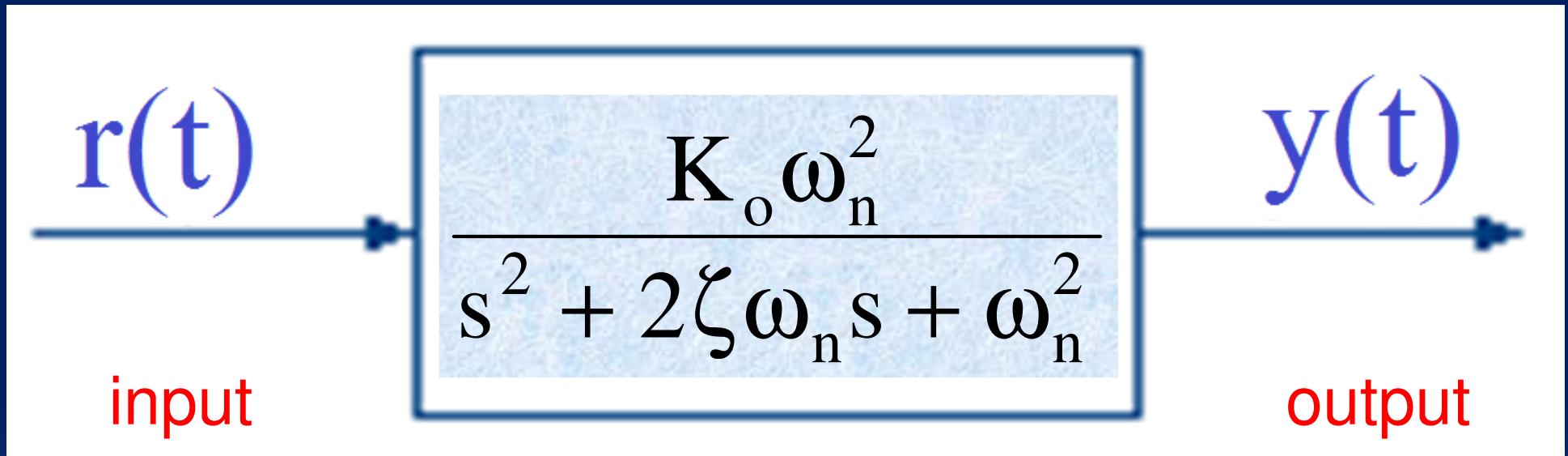
$$\left\{ \begin{array}{l} \Delta > 0 \rightarrow (\zeta^2 - 1) > 0 \rightarrow \zeta^2 > 1 \rightarrow \zeta > 1 \\ \Delta = 0 \rightarrow (\zeta^2 - 1) = 0 \rightarrow \zeta^2 = 1 \rightarrow \zeta = 1 \\ \Delta < 0 \rightarrow (\zeta^2 - 1) < 0 \rightarrow \zeta^2 < 1 \rightarrow \zeta < 1 \end{array} \right.$$

$\zeta > 1 \rightarrow$  poles are real and distinct

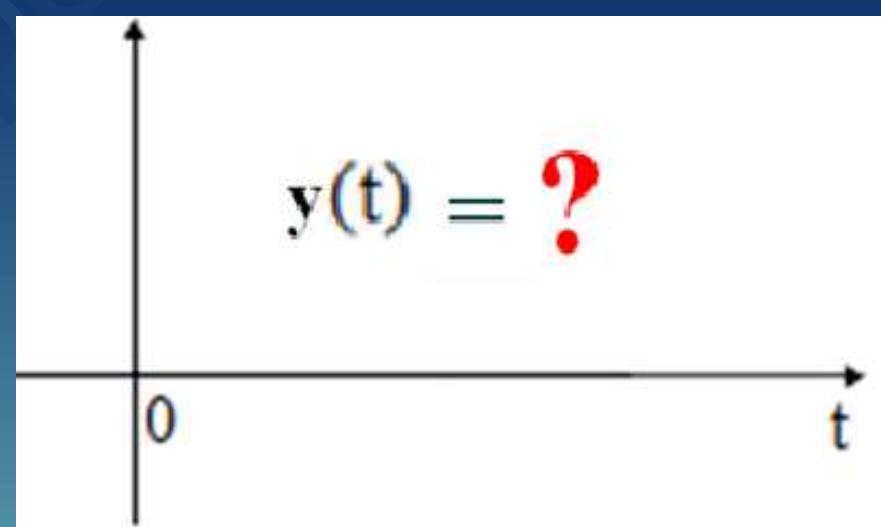
$\zeta = 1 \rightarrow$  poles are real and repeated

$0 < \zeta < 1 \rightarrow$  poles are complex conjugate

# Time domain analysis - 2<sup>nd</sup> order systems

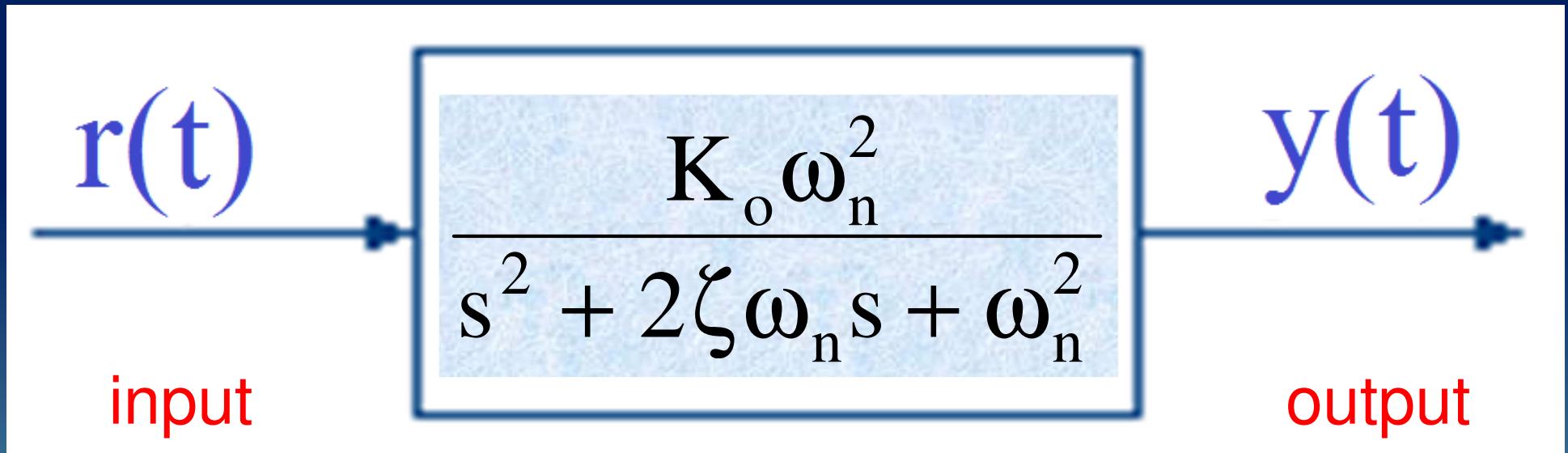


# unit step input



# What is the output? *(step response)*

## Time domain analysis - 2<sup>nd</sup> order systems

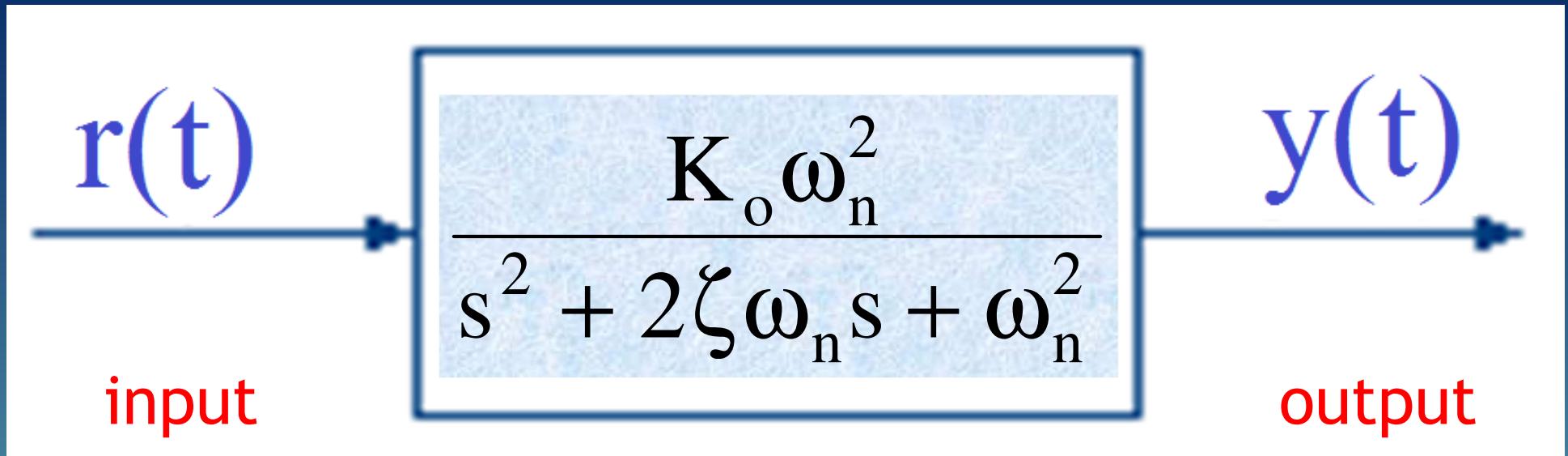


$$Y(s) = \frac{K_o \omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot R(s)$$

and since  $r(t) = \text{unit step}$ :

$$Y(s) = \frac{K_o \omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot \frac{1}{s}$$

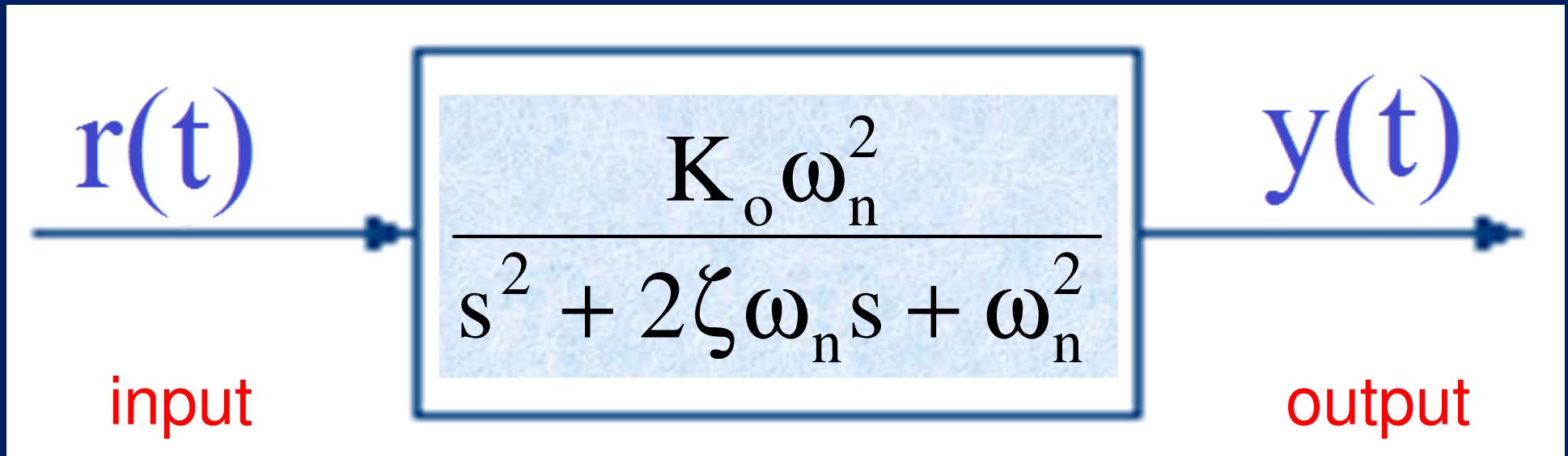
# Time domain analysis - 2<sup>nd</sup> order systems



$$y(t) = \mathcal{L}^{-1}[Y(s)]$$

the **unit step** response depends on the value of  $\zeta$ :

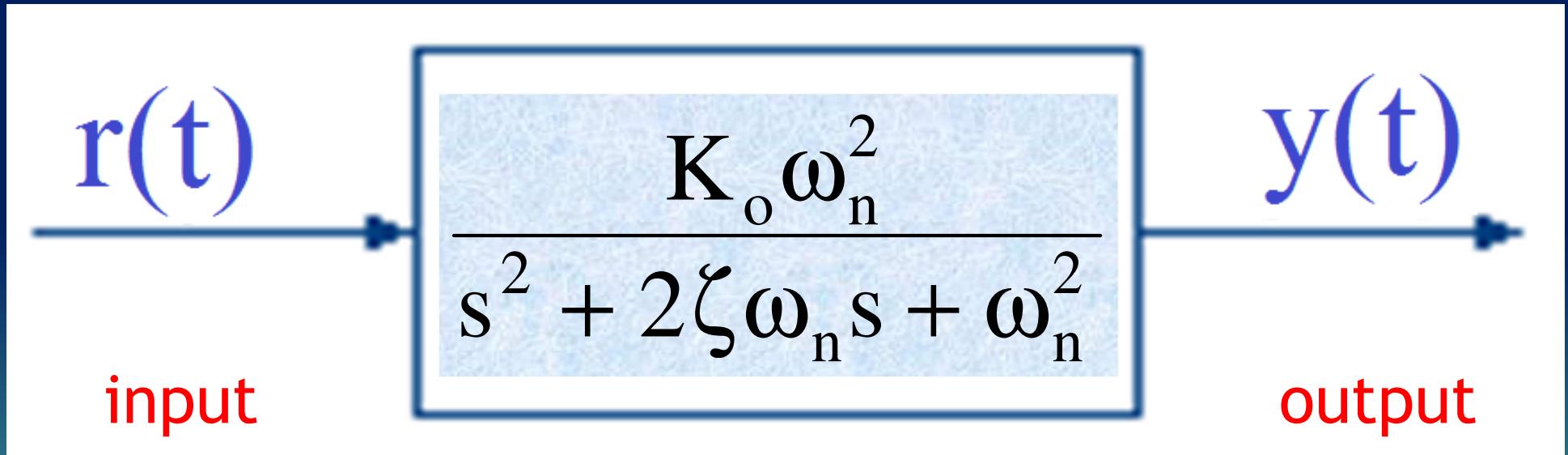
- a)  $0 < \zeta < 1$  (*under damping*)
  - b)  $\zeta = 1$  (*critical damping*)
  - c)  $\zeta > 1$  (*over damping*)



Let us now see  $y(t)$ , the outputs to the unit step in these 3 cases, starting with case (a)

a)  $0 < \zeta < 1$  (*under damping*)

## Time domain analysis - 2<sup>nd</sup> order systems



$$y(t) = \mathcal{L}^{-1}[Y(s)]$$

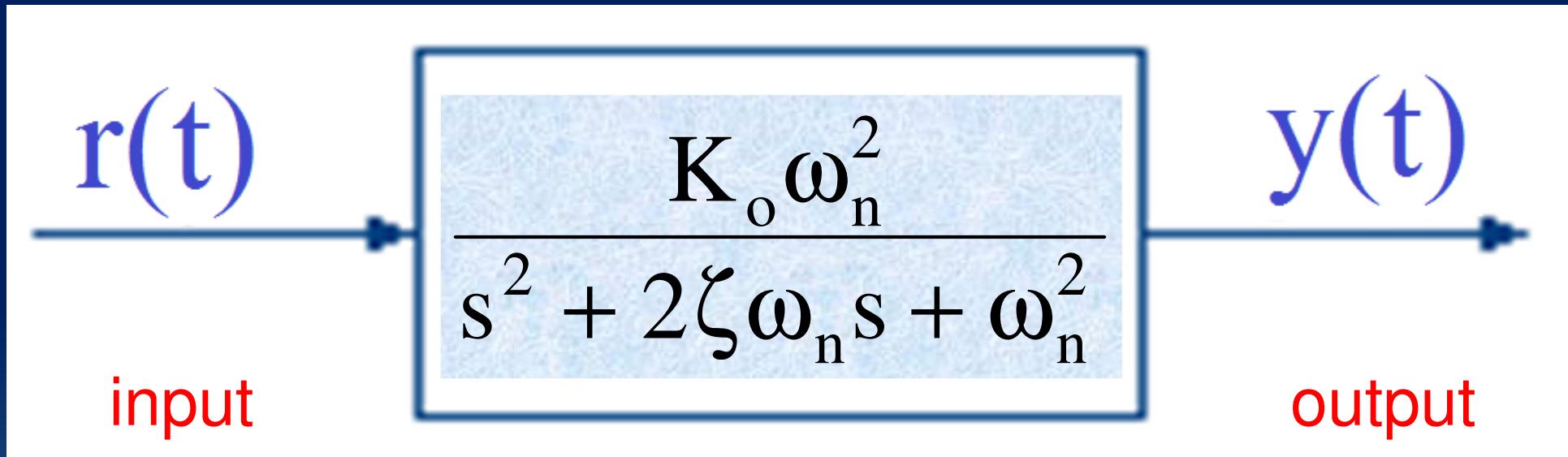
So, in the case  $0 < \zeta < 1$  (*under damping*) the **unit step response** is:

$$y(t) = K_o \left[ 1 - e^{-\zeta\omega_n t} \left( \cos \omega_d t + \frac{\zeta}{\sqrt{1-\zeta^2}} \cdot \sin \omega_d t \right) \right], \quad t > 0$$

where

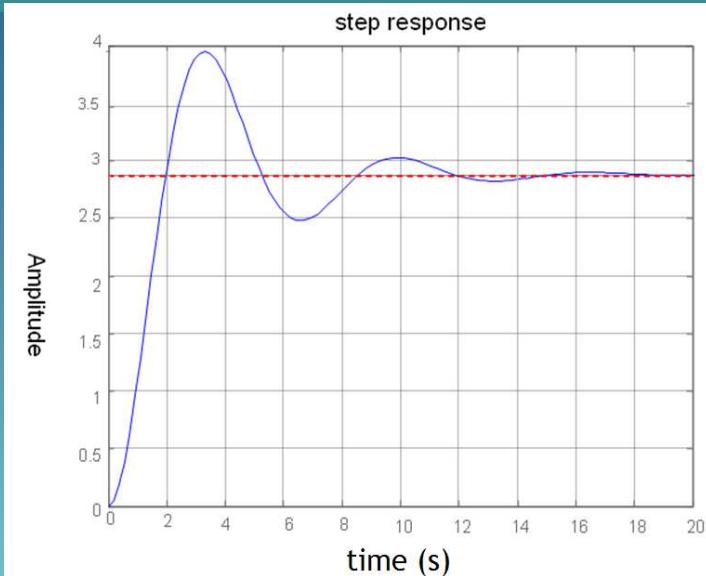
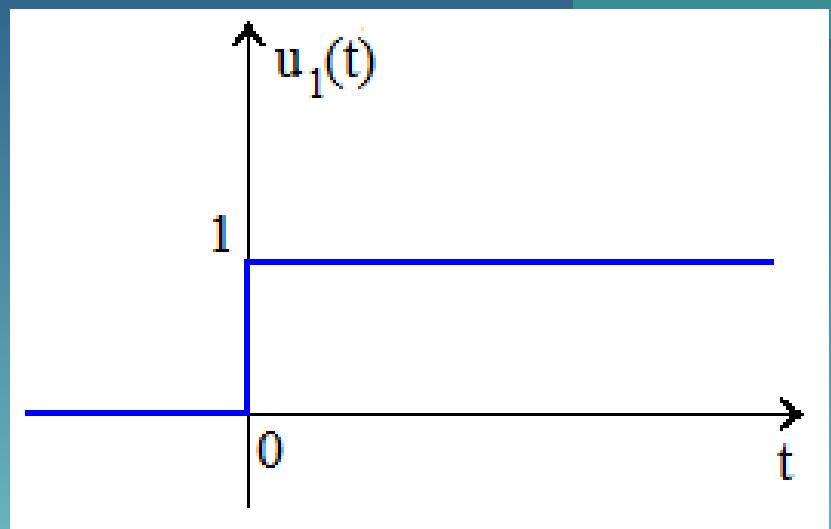
$$\omega_d = \omega_n \cdot \sqrt{1 - \zeta^2} \quad (\textit{damping frequency})$$

## Time domain analysis - 2<sup>nd</sup> order systems



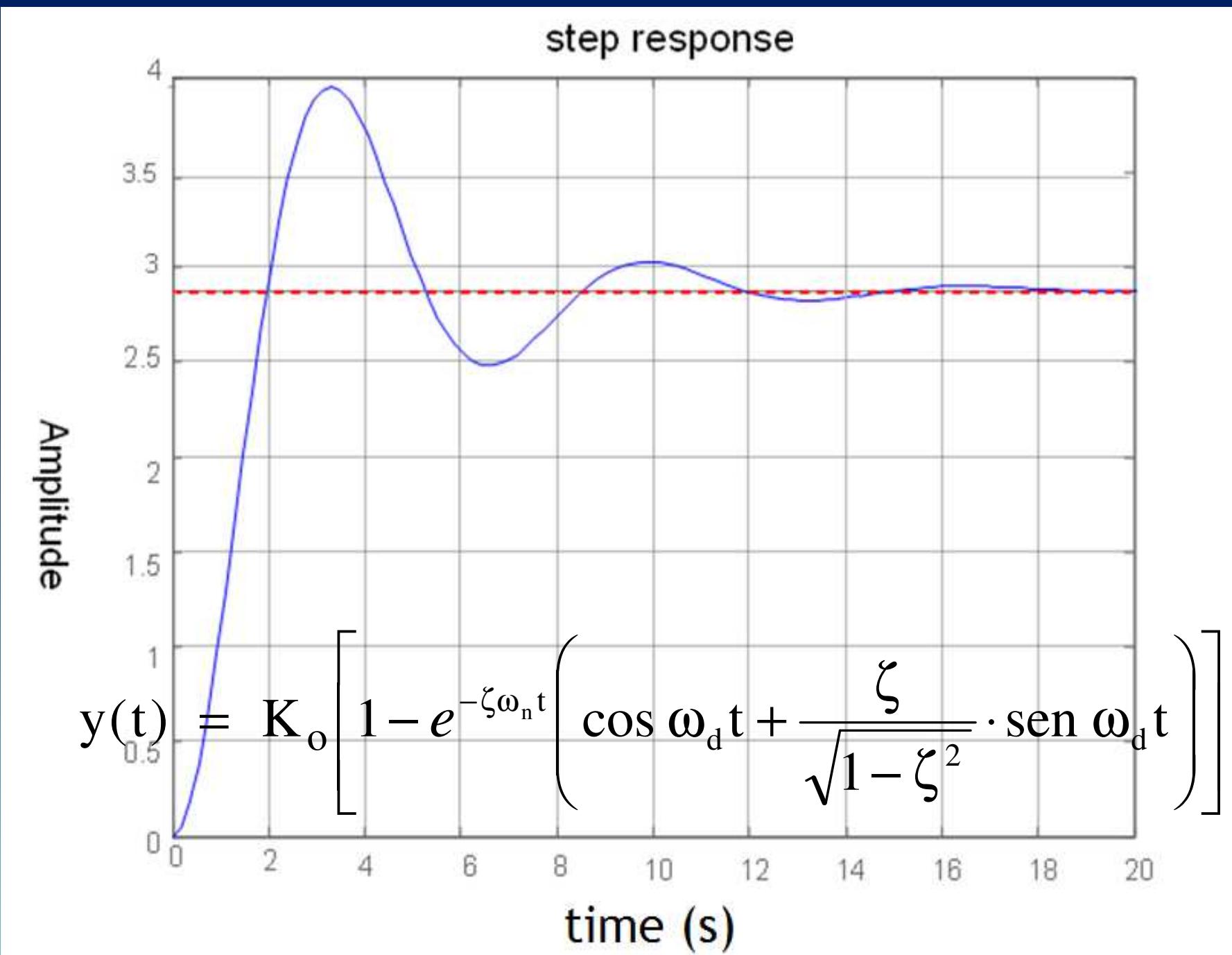
unit step response :

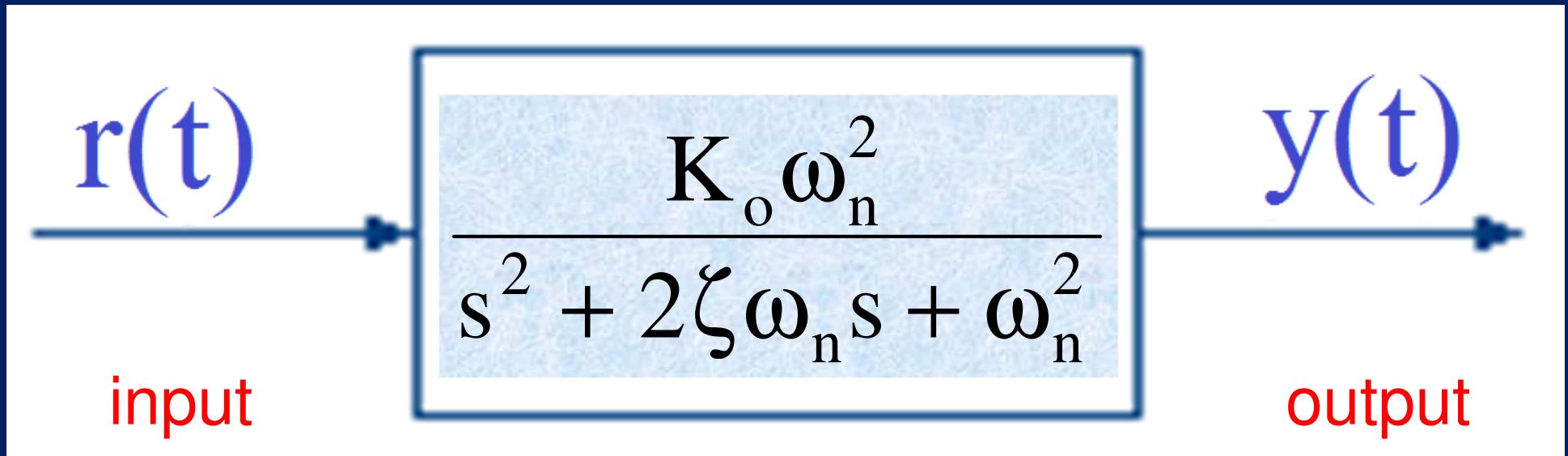
$$y(t) = K_o \left[ 1 - e^{-\zeta\omega_n t} \left( \cos \omega_d t + \frac{\zeta}{\sqrt{1-\zeta^2}} \cdot \sin \omega_d t \right) \right]$$



# Time domain analysis - 2<sup>nd</sup> order systems

unit step response:

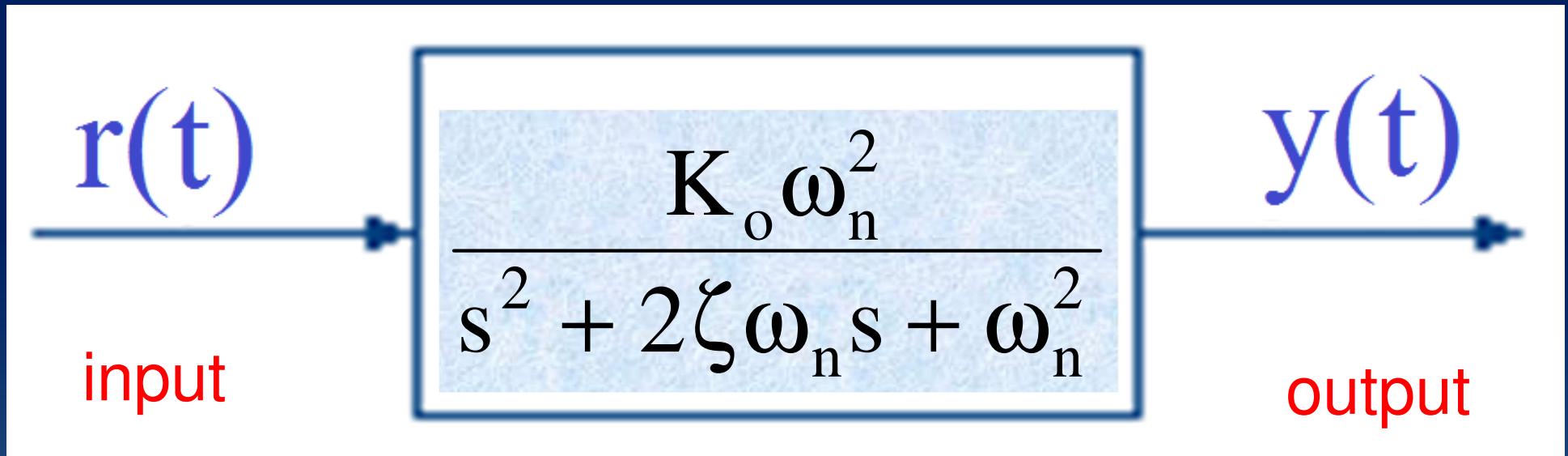




Let us now see  $y(t)$ , the output to the **unit step** for case (b)

b)  $\zeta = 1$  (*critical damping*)

## Time domain analysis - 2<sup>nd</sup> order systems

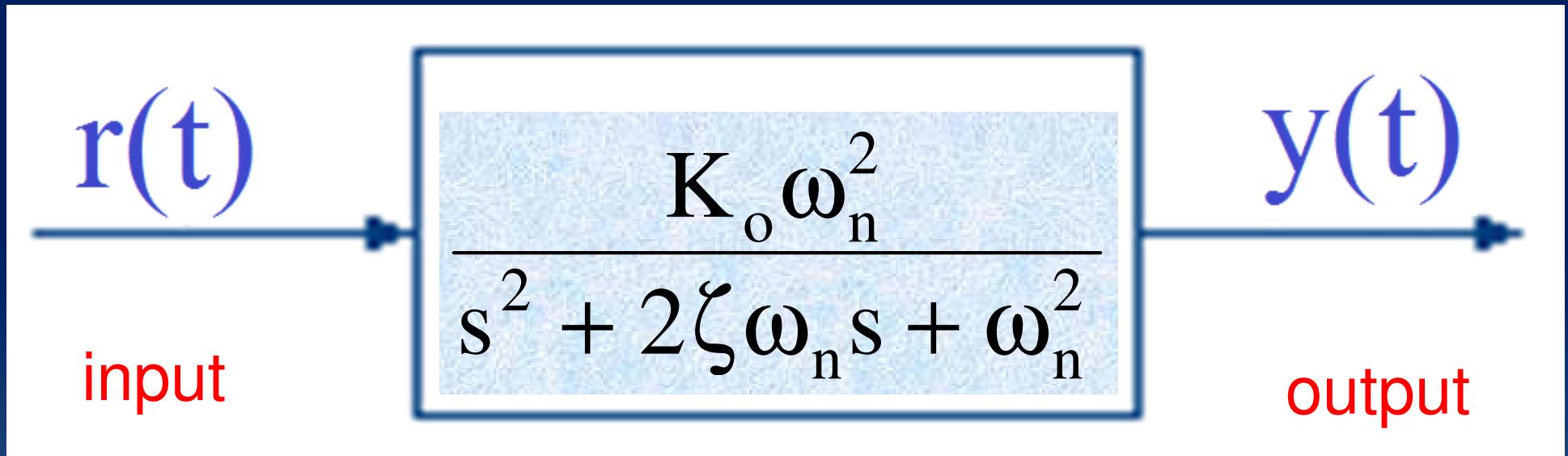


$$y(t) = \mathcal{L}^{-1}[Y(s)]$$

In the case  $\zeta = 1$  (*critical damping*) the unit step response is:

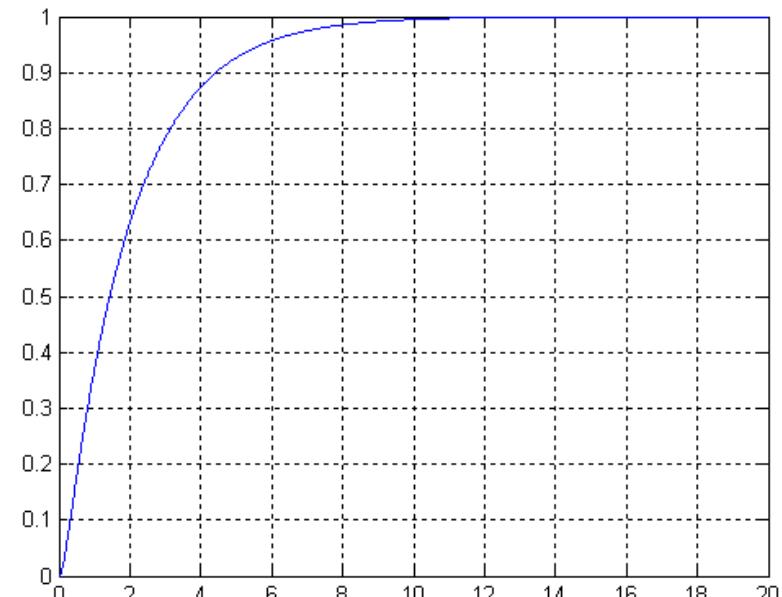
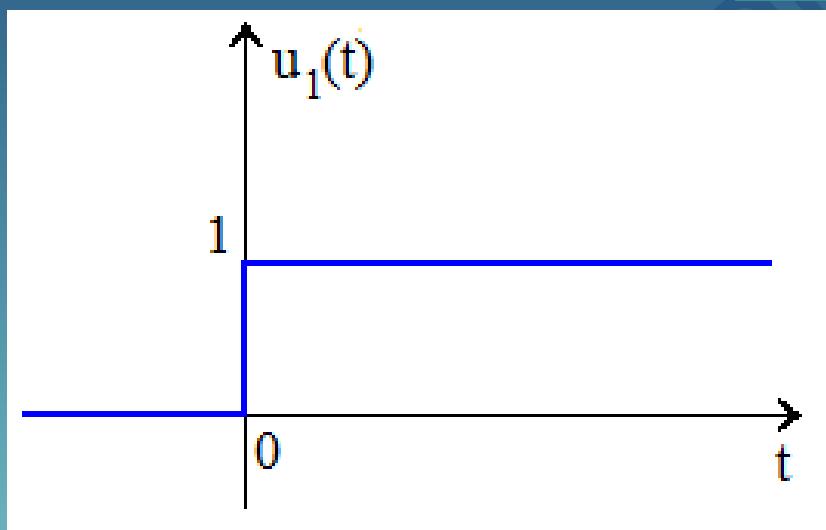
$$y(t) = K_o [ 1 - e^{-\zeta\omega_n t} \cdot (1 + \omega_n t) ], \quad t > 0$$

# Time domain analysis - 2<sup>nd</sup> order systems



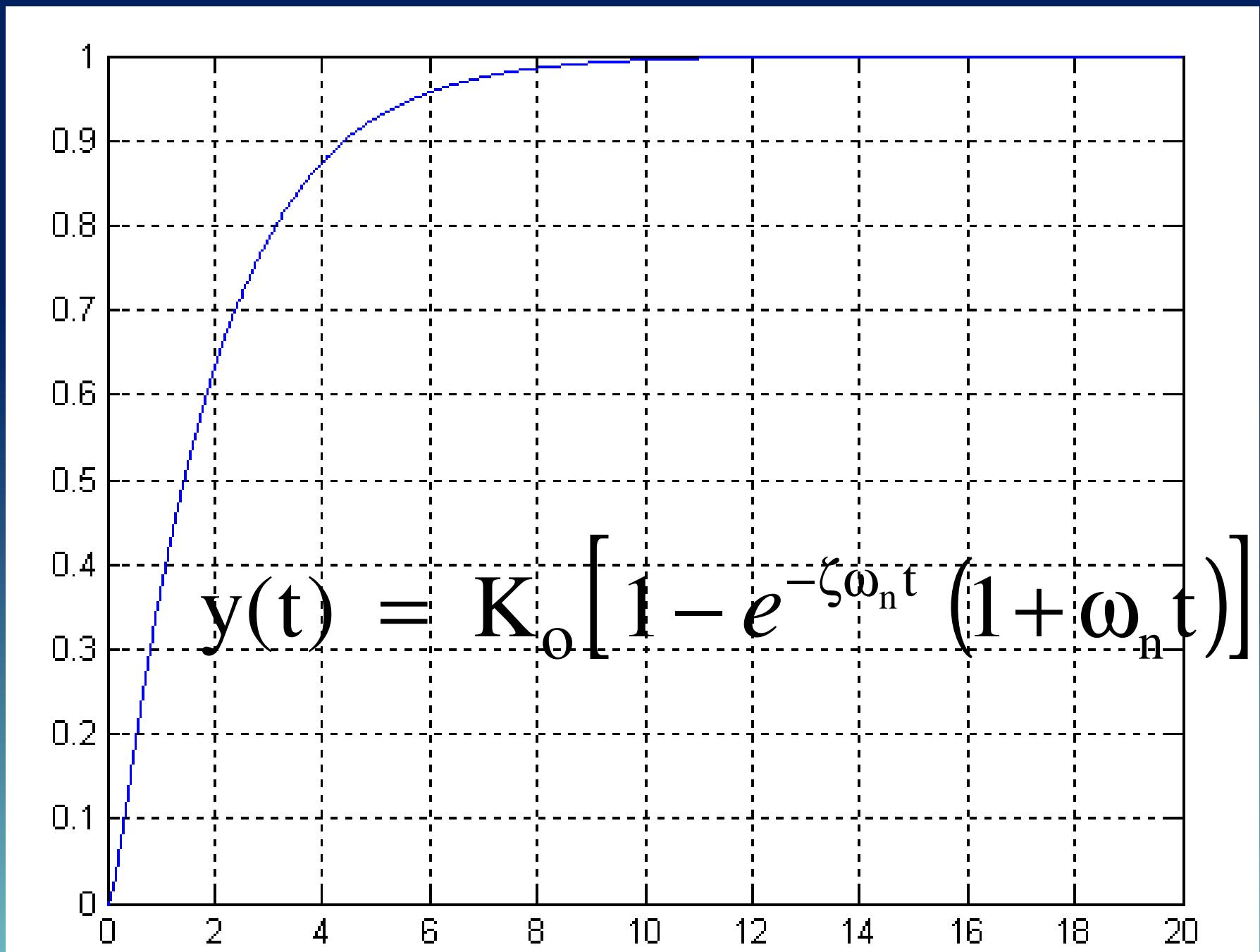
## unit step response :

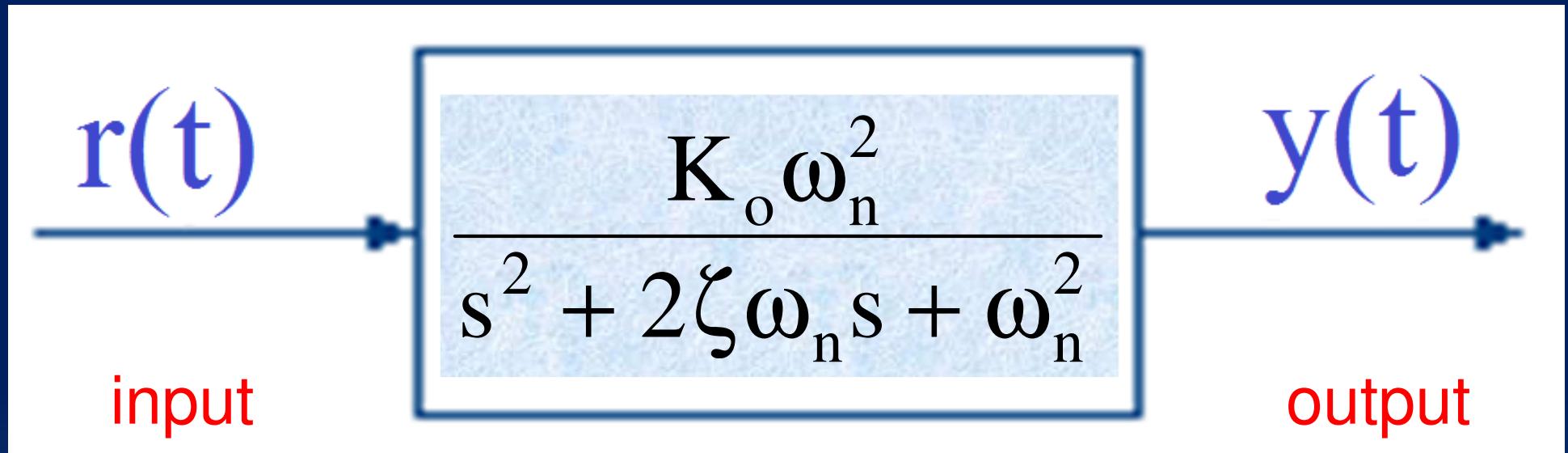
$$y(t) = K_o \left[ 1 - e^{-\zeta \omega_n t} (1 + \omega_n t) \right]$$



# Time domain analysis - 2<sup>nd</sup> order systems

unit step response :

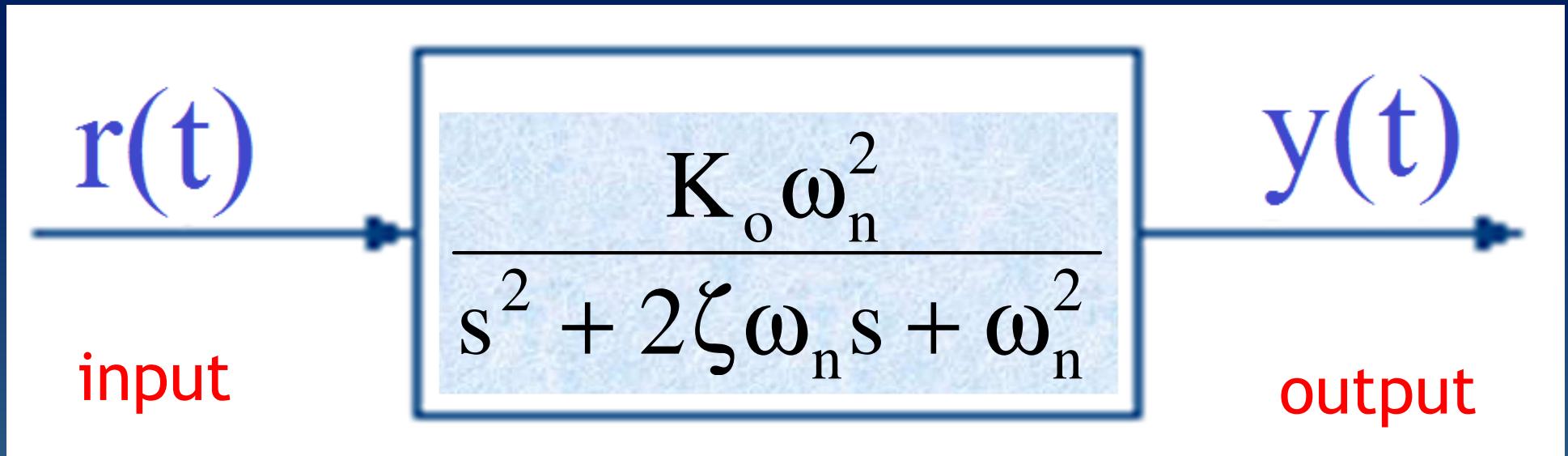




Finally, let us now see  $y(t)$ , the output to the **unit step** for case (c)

c)  $\zeta > 1$  (*over damping*)

## Time domain analysis - 2<sup>nd</sup> order systems



$$y(t) = \mathcal{L}^{-1}[Y(s)]$$

In the case  $\zeta > 1$  (*over damping*) the **unit step response** is:

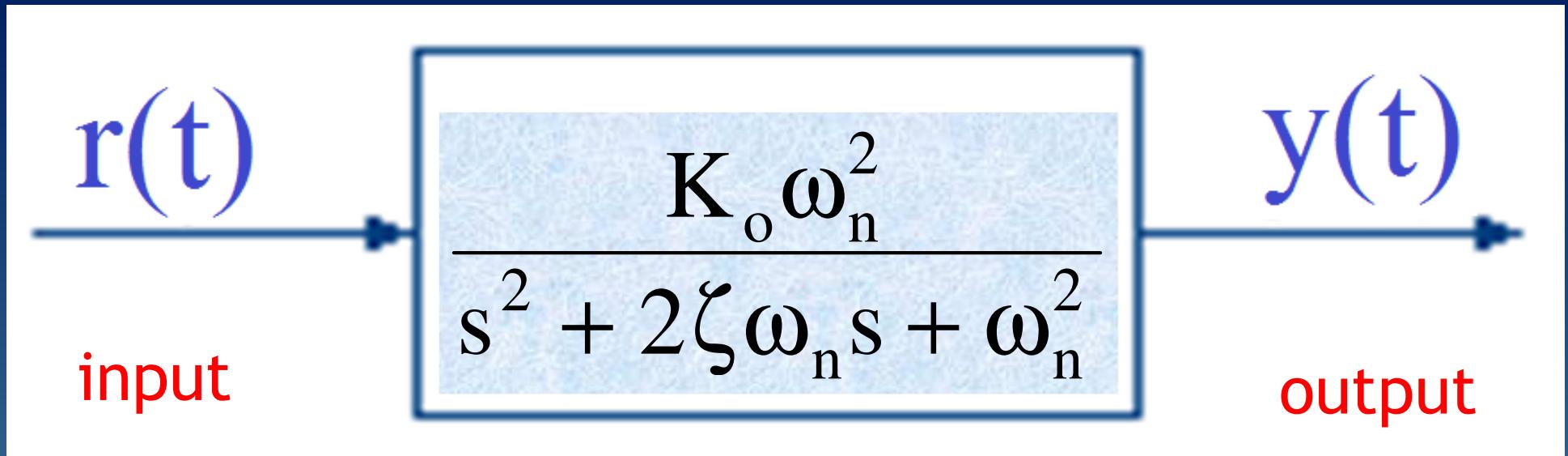
$$y(t) = K_o \left[ 1 + \frac{\omega_n}{2\sqrt{\zeta^2 - 1}} \cdot \left( \frac{e^{p_1 t}}{p_1} - \frac{e^{p_2 t}}{p_2} \right) \right], \quad t > 0$$

where

$$p_{1,2} = -\zeta\omega_n \mp \omega_n \sqrt{\zeta^2 - 1} = -\omega_n (\zeta \pm \sqrt{\zeta^2 - 1})$$

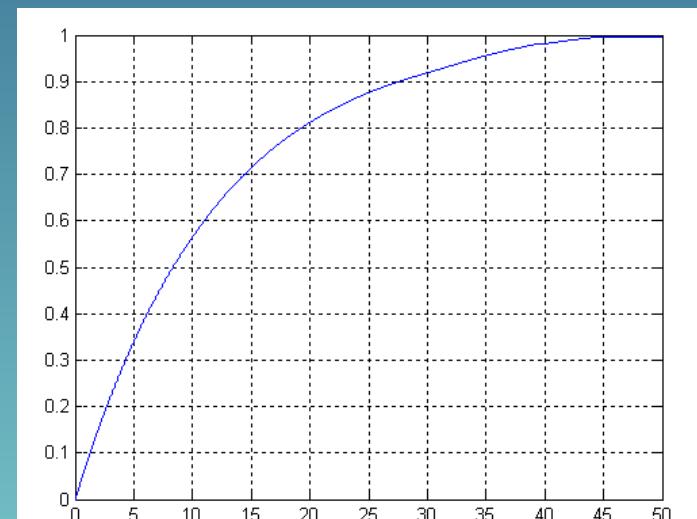
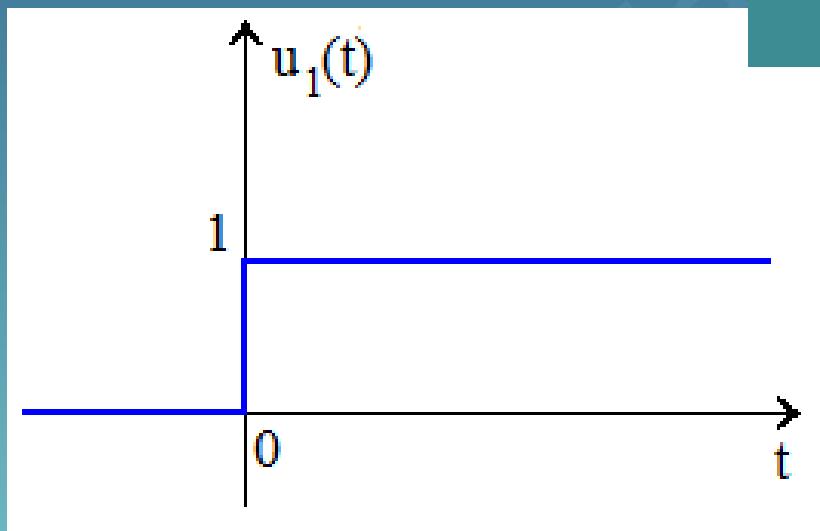
*System has real poles*

## Time domain analysis - 2<sup>nd</sup> order systems



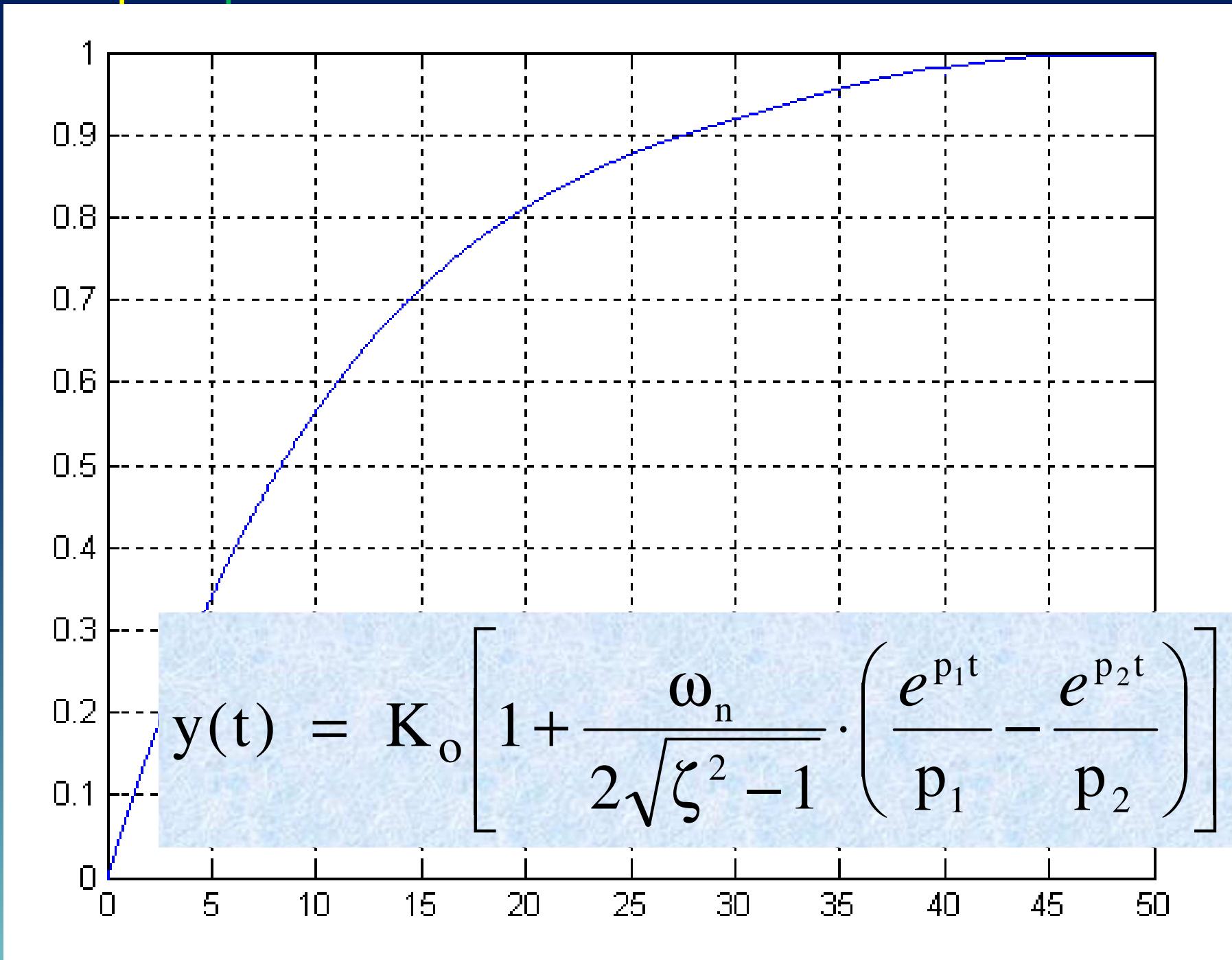
unit step response:

$$y(t) = K_o \left[ 1 + \frac{\omega_n}{2\sqrt{\zeta^2 - 1}} \cdot \left( \frac{e^{p_1 t}}{p_1} - \frac{e^{p_2 t}}{p_2} \right) \right]$$

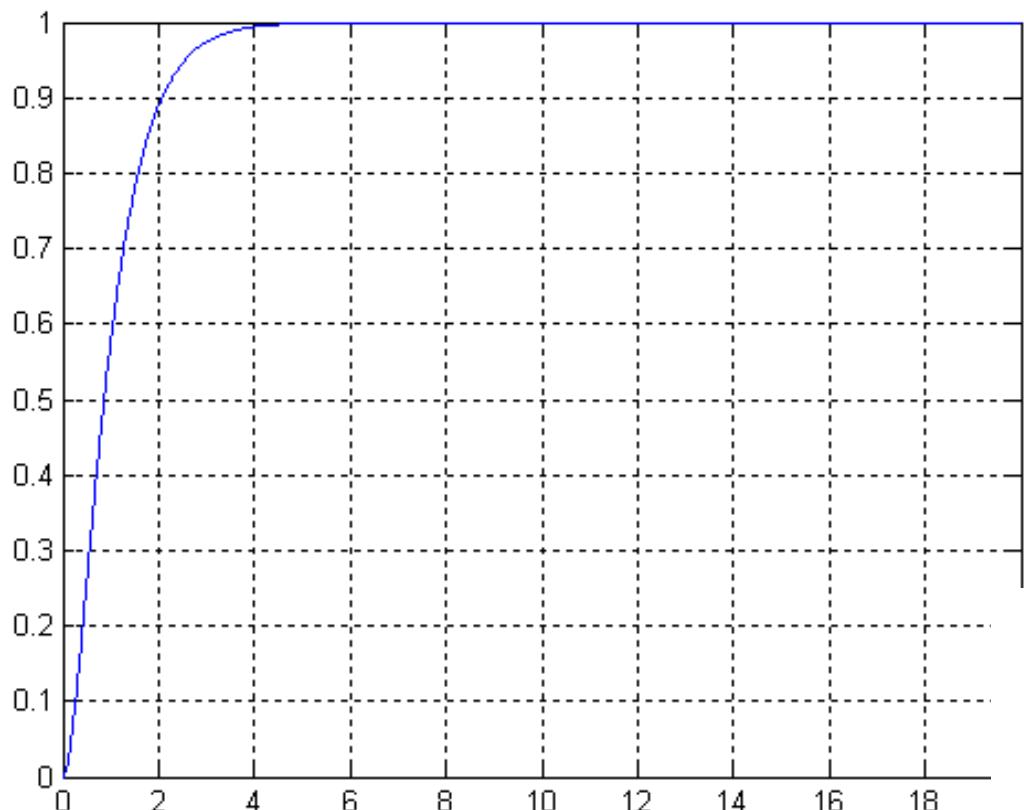


# Time domain analysis - 2<sup>nd</sup> order systems

unit step response:



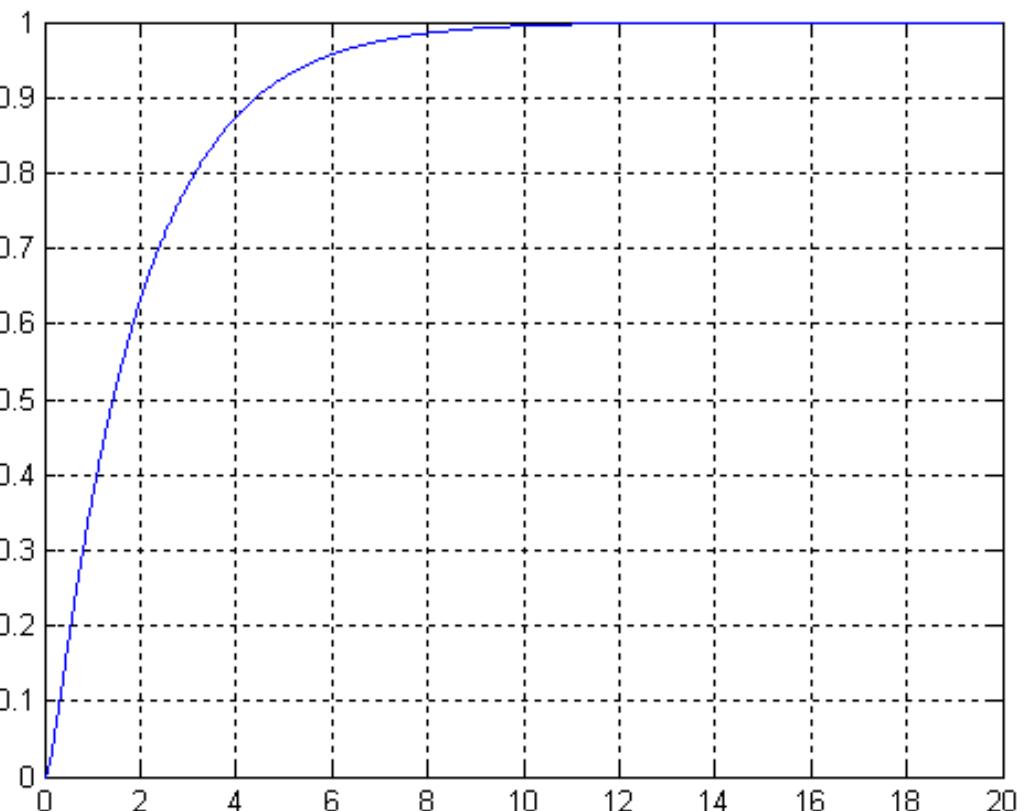
## Time domain analysis - 2<sup>nd</sup> order systems



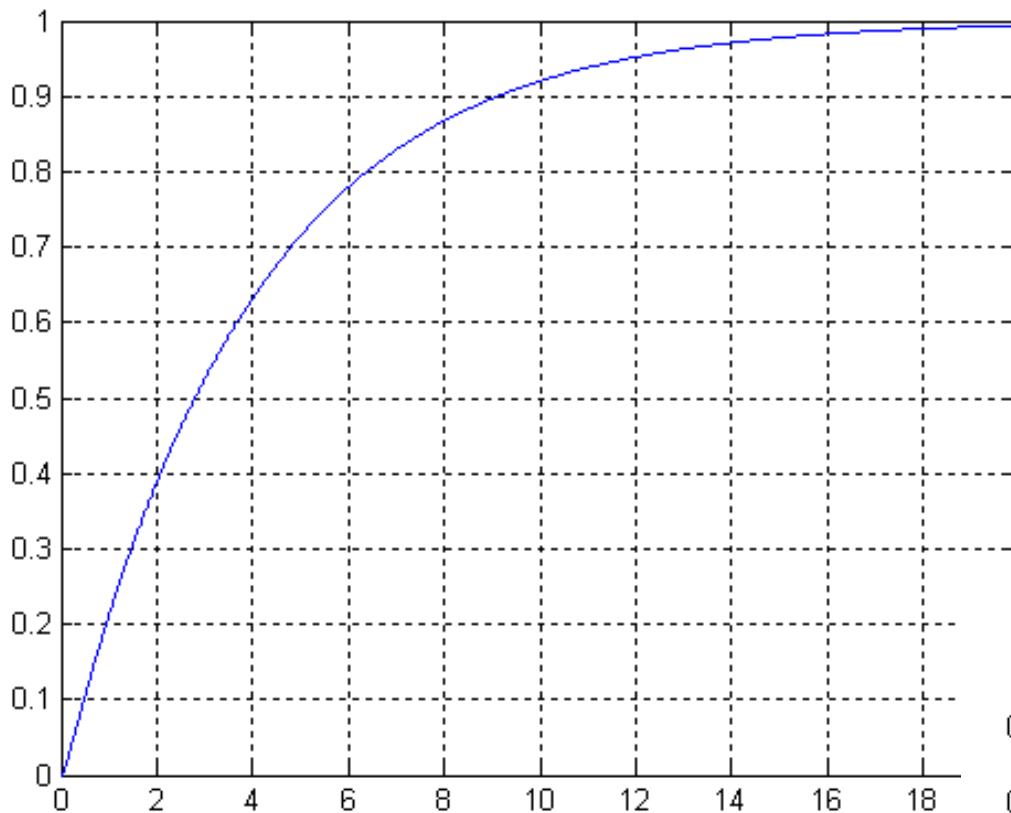
$\zeta = 1$

$\zeta = 2$

unit step  
response



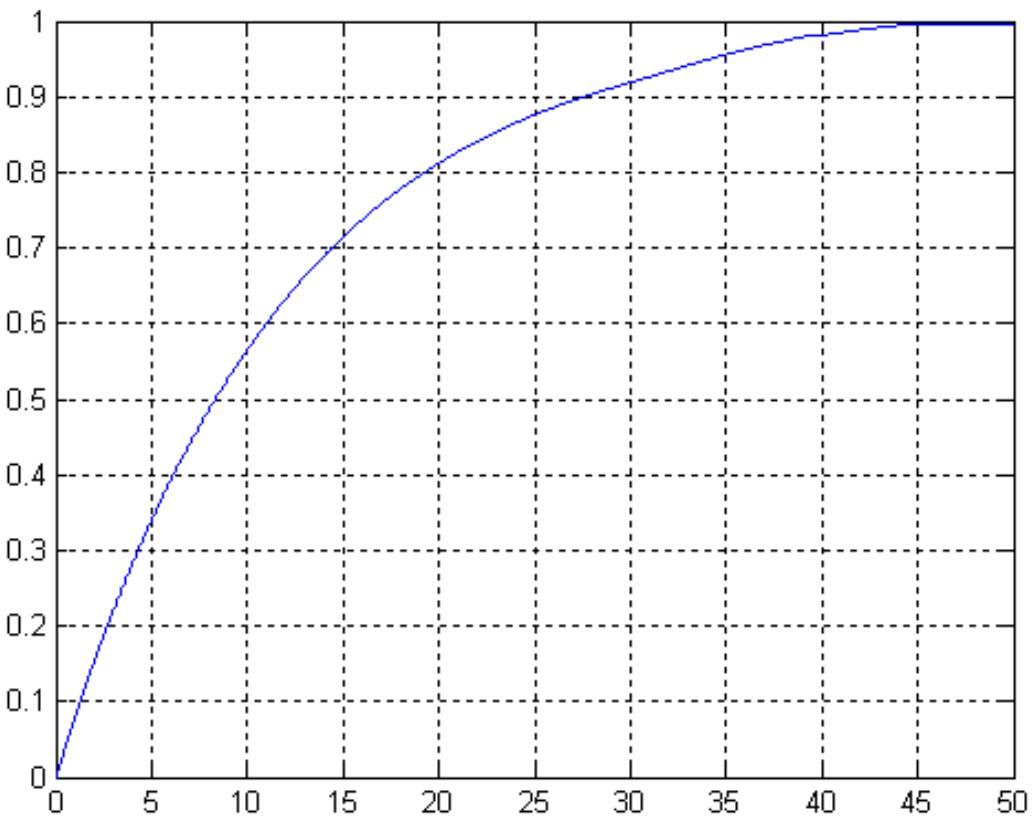
## Time domain analysis - 2<sup>nd</sup> order systems



$\zeta = 4$

$\zeta = 12$

unit step  
response



## Time domain analysis - 2<sup>nd</sup> order systems

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Let us now analyse some parameters associated with the under damping case

$$0 < \zeta < 1 \text{ (under damping)}$$

In the case  $0 < \zeta < 1$ ,  
the unit step response

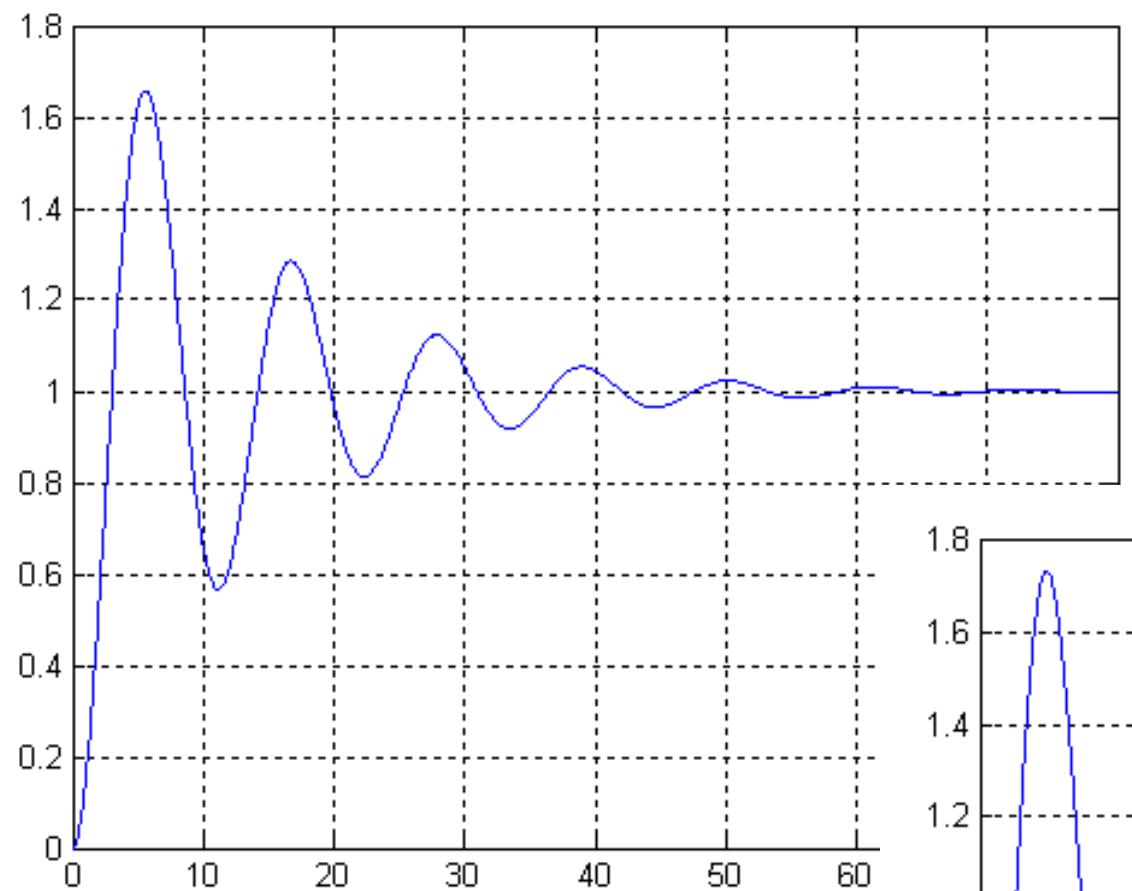
$$y(t) = K_o \left[ 1 - e^{-\zeta \omega_n t} \left( \cos \omega_d t + \frac{\zeta}{\sqrt{1-\zeta^2}} \cdot \sin \omega_d t \right) \right], \quad t > 0$$

can have many different forms, depending on the  
values of  $\zeta$  (*damping coefficient*),  
 $\omega_n$  (*natural frequency*) e  $K_o$  (*gain*)

Observe that  $\omega_d$  depends on  $\zeta$  and  $\omega_n$

$$\omega_d = \omega_n \cdot \sqrt{1 - \zeta^2} \quad (\textit{damping frequency})$$

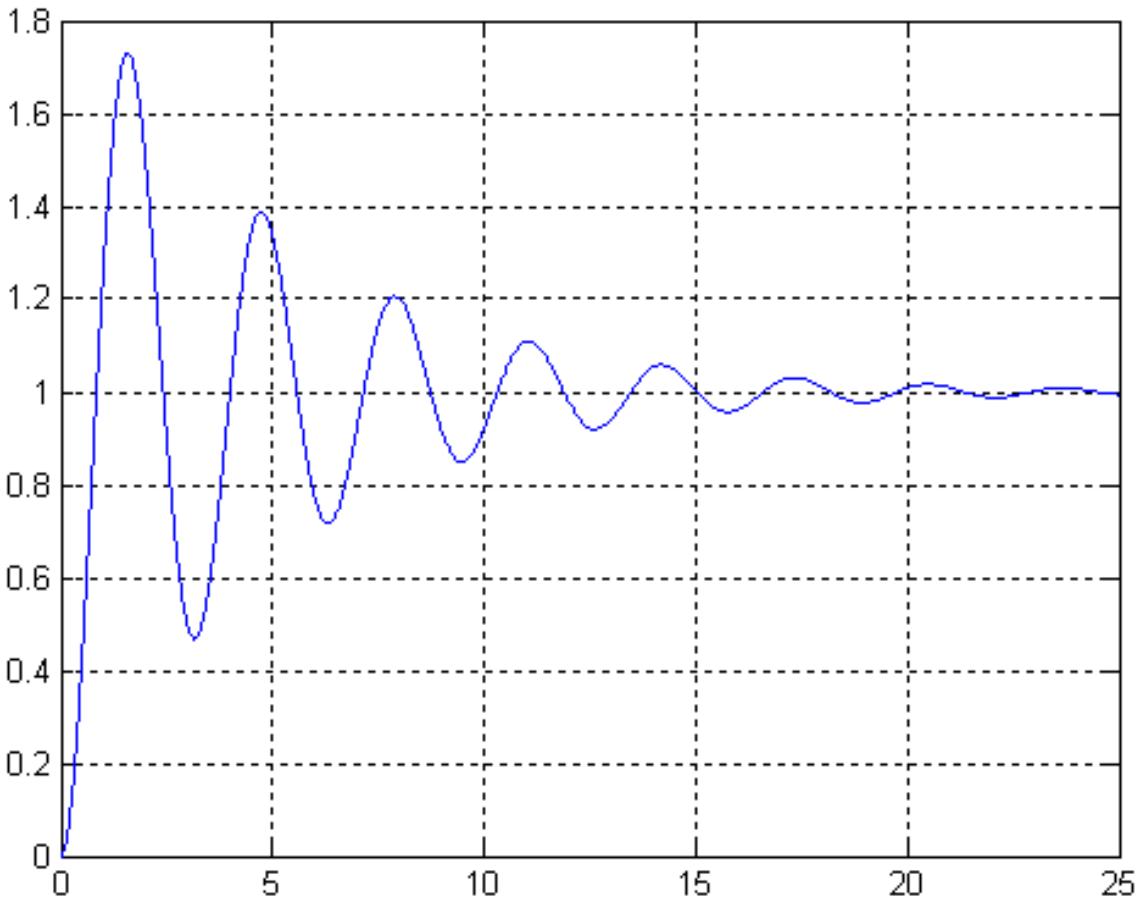
## Time domain analysis - 2<sup>nd</sup> order systems



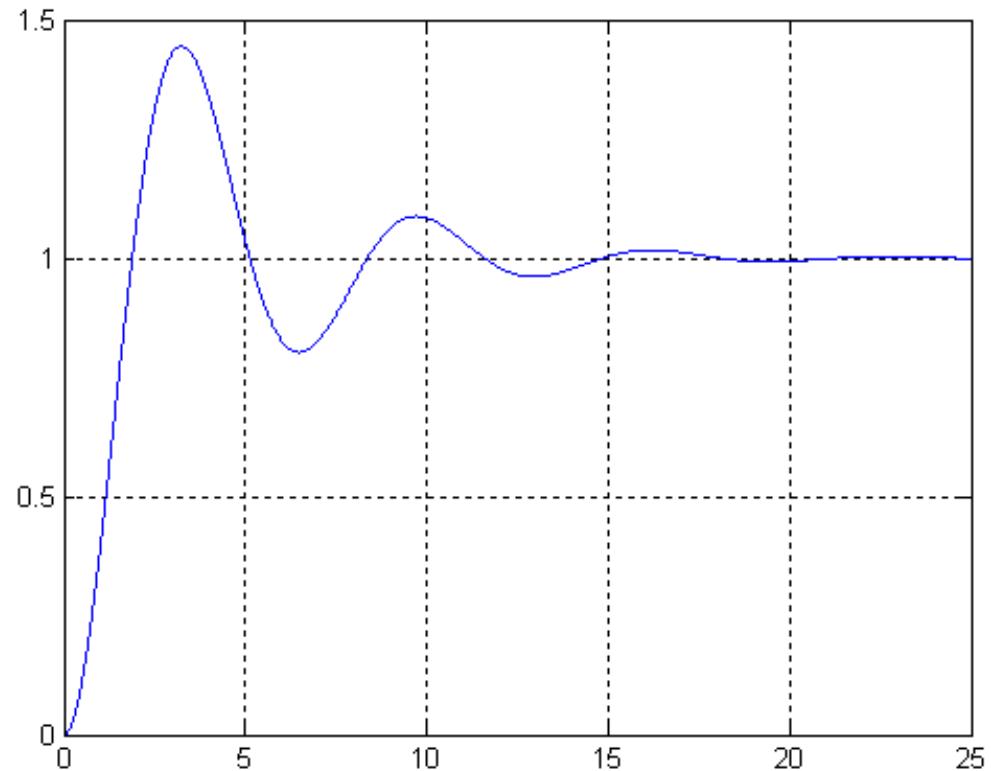
$$\zeta = 0,132$$
$$\omega_n = 0,57$$

$$\zeta = 0,1$$
$$\omega_n = 2$$

unit step  
response



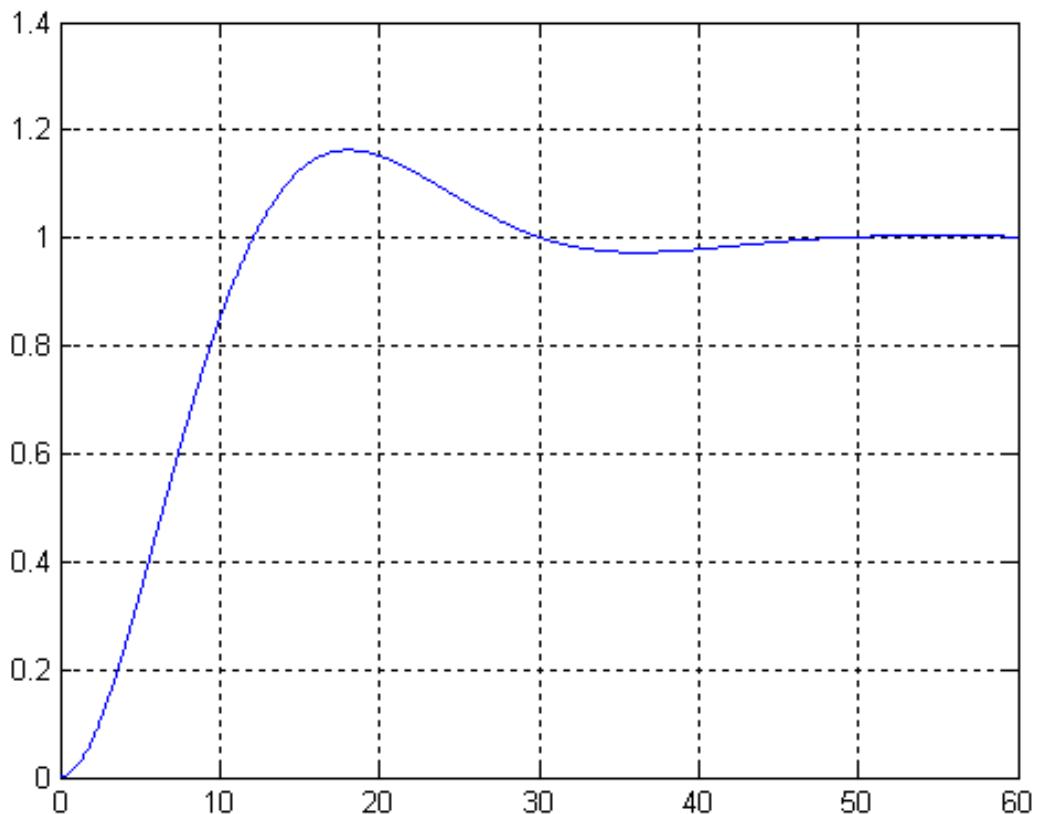
## Time domain analysis - 2<sup>nd</sup> order systems



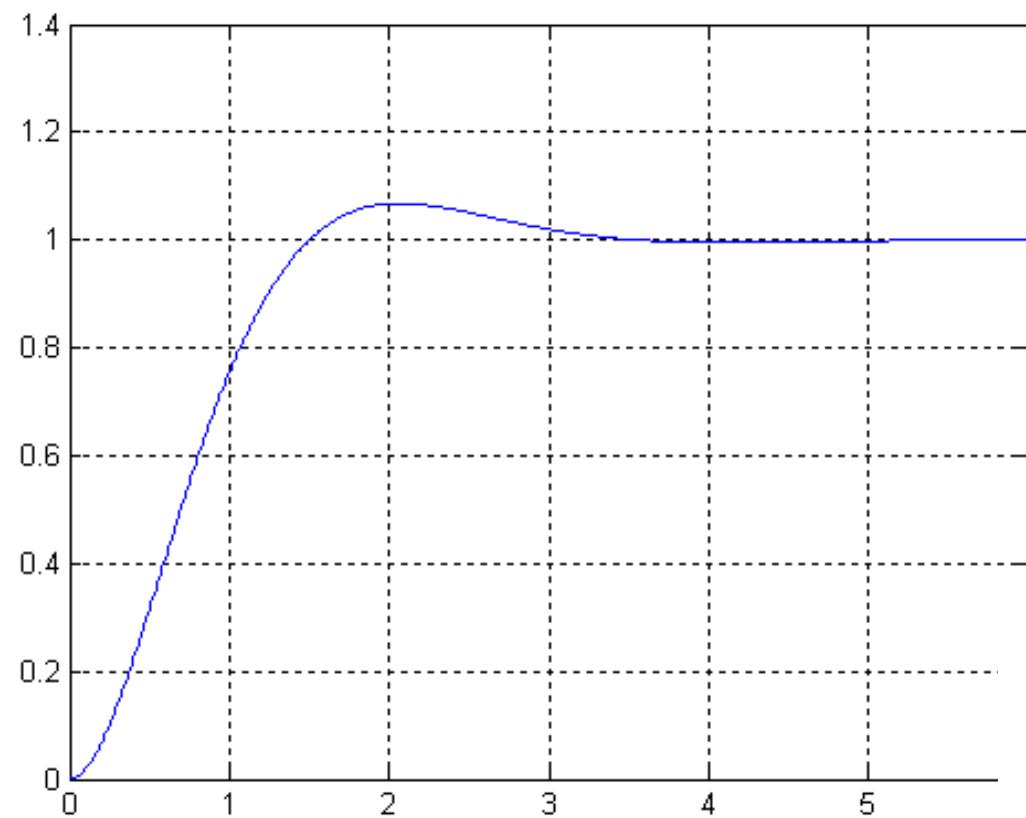
$$\zeta = 0,25$$
$$\omega_n = 1$$

$$\zeta = 0,5$$
$$\omega_n = 0,2$$

unit step  
response



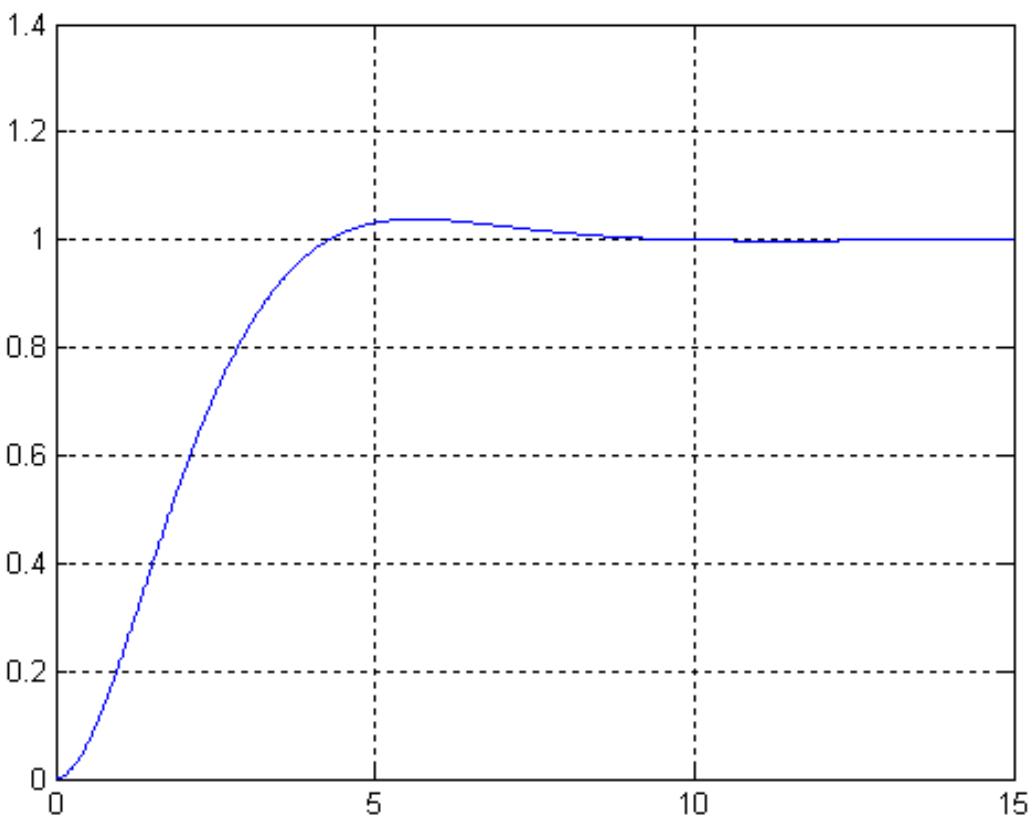
## Time domain analysis - 2<sup>nd</sup> order systems



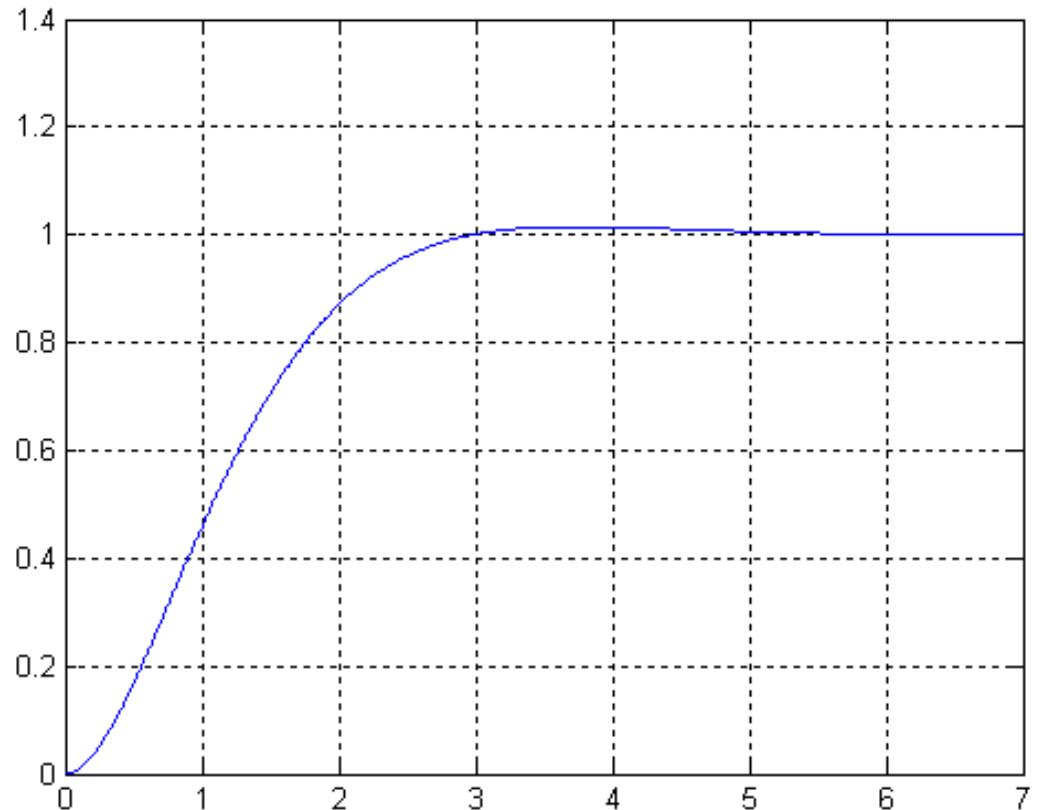
$$\zeta = 0,65$$
$$\omega_n = 2$$

$$\zeta = 0,72$$
$$\omega_n = 0,8$$

unit step  
response



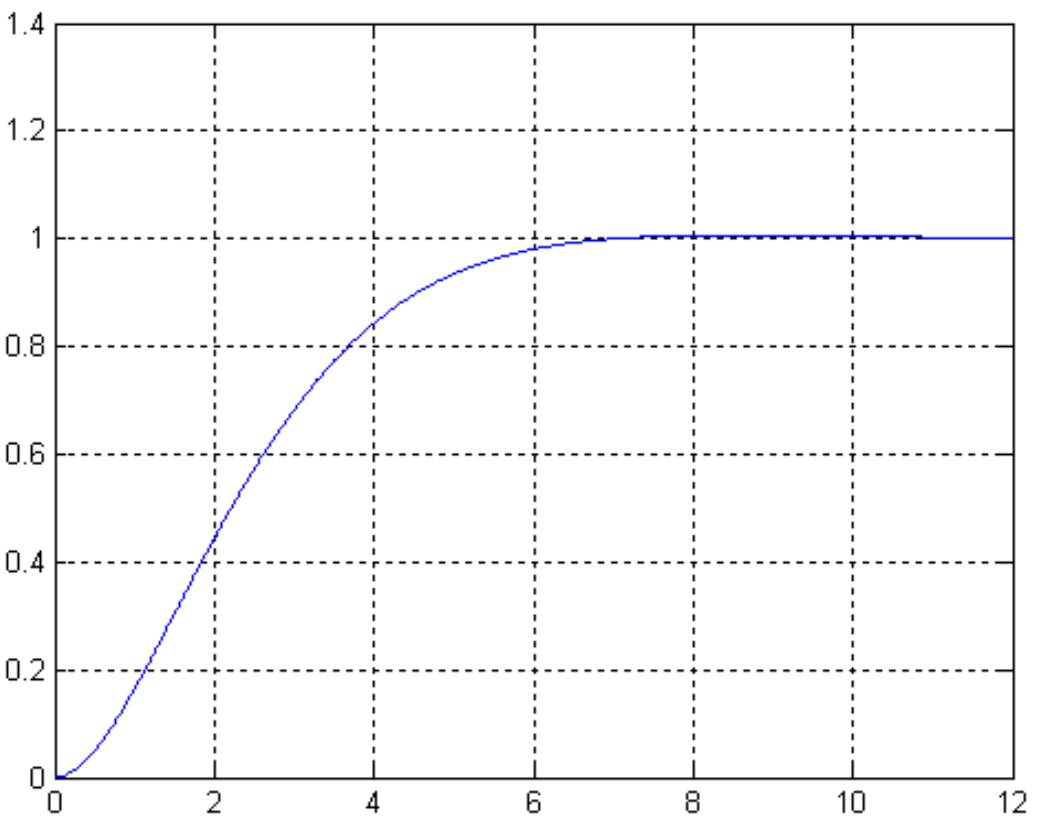
## Time domain analysis - 2<sup>nd</sup> order systems



$$\zeta = 0,8$$
$$\omega_n = 1,4$$

$$\zeta = 0,85$$
$$\omega_n = 0,7$$

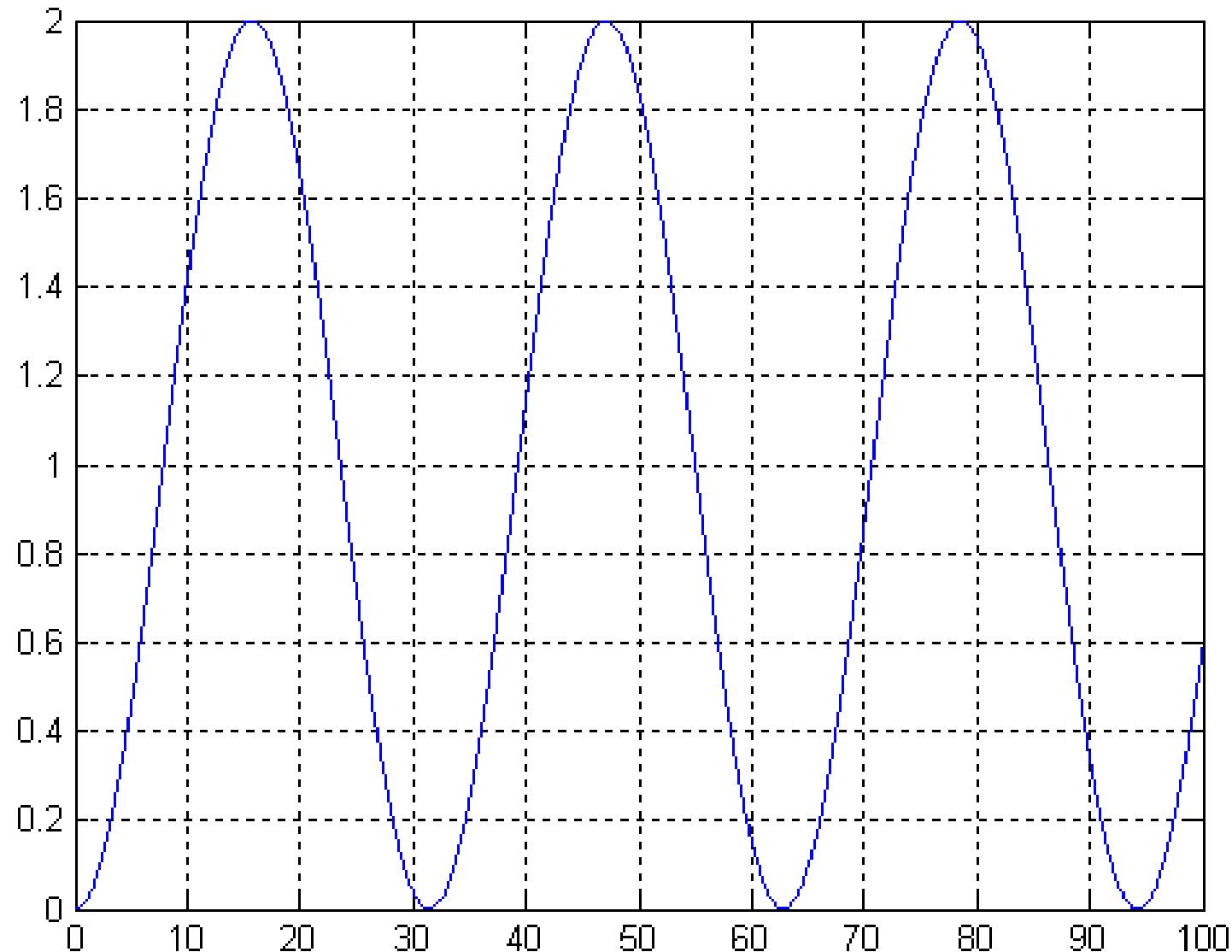
unit step  
response



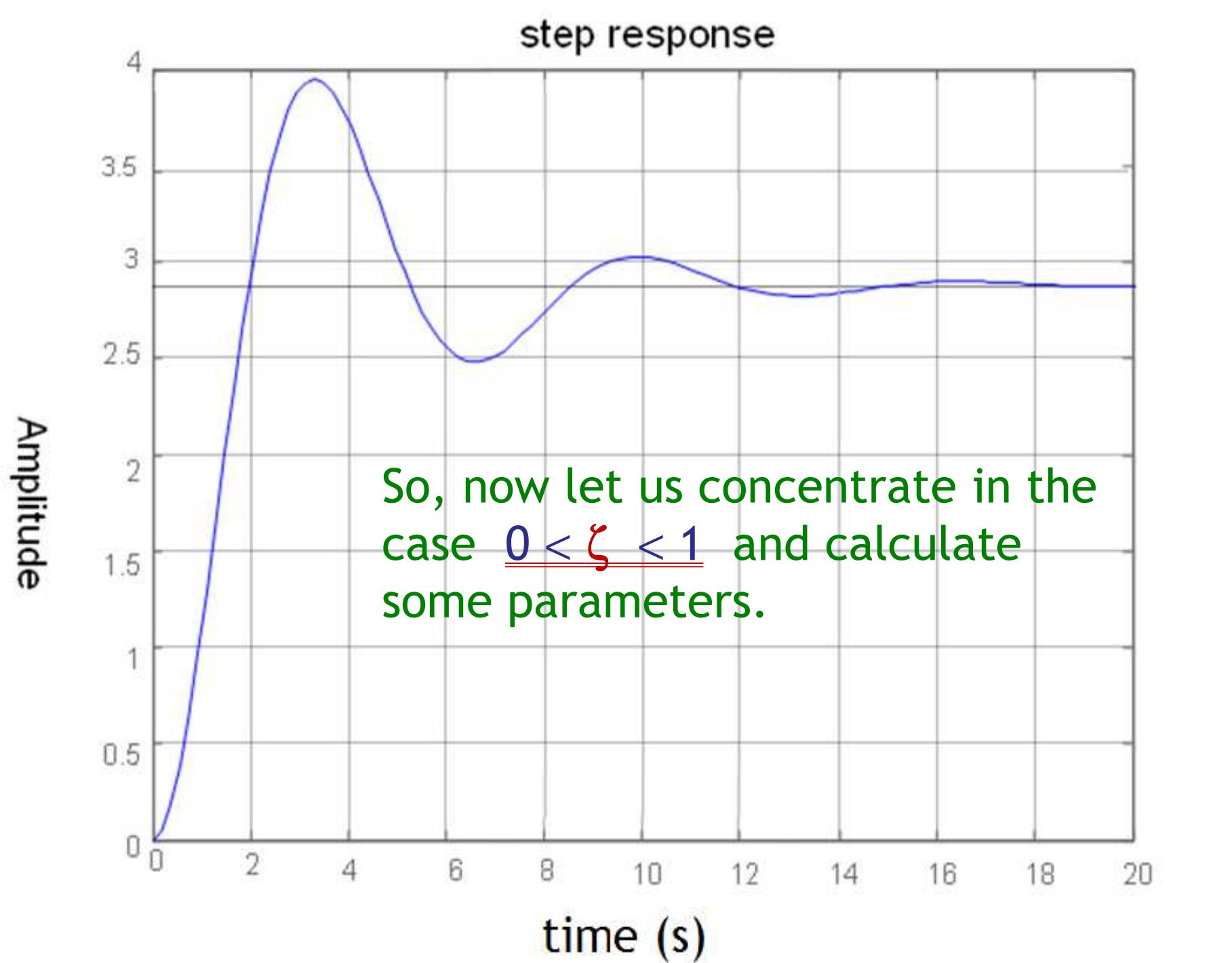
# Time domain analysis - 2<sup>nd</sup> order systems

unit step  
response

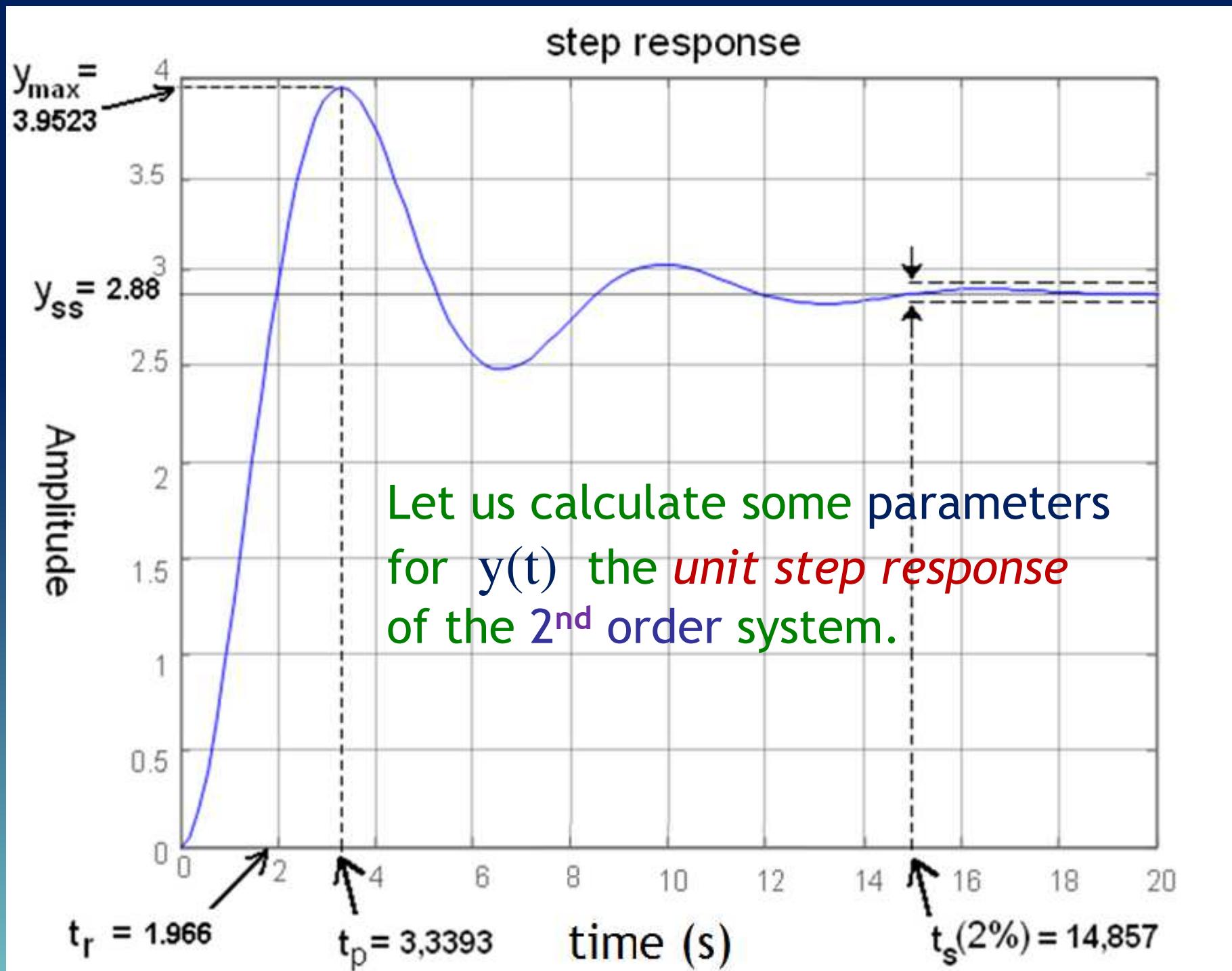
$$\zeta = 0 \\ \omega_n = 0,2$$



## Time domain analysis - 2<sup>nd</sup> order systems



## Time domain analysis - 2<sup>nd</sup> order systems



*steady state output*

$y_{ss}$

$y_{ss}$  = steady state response  
 ( *steady state output* )

$$Y(s) = \frac{K_o \omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot R(s) \xrightarrow{\text{R}(s) = \frac{1}{s}}$$

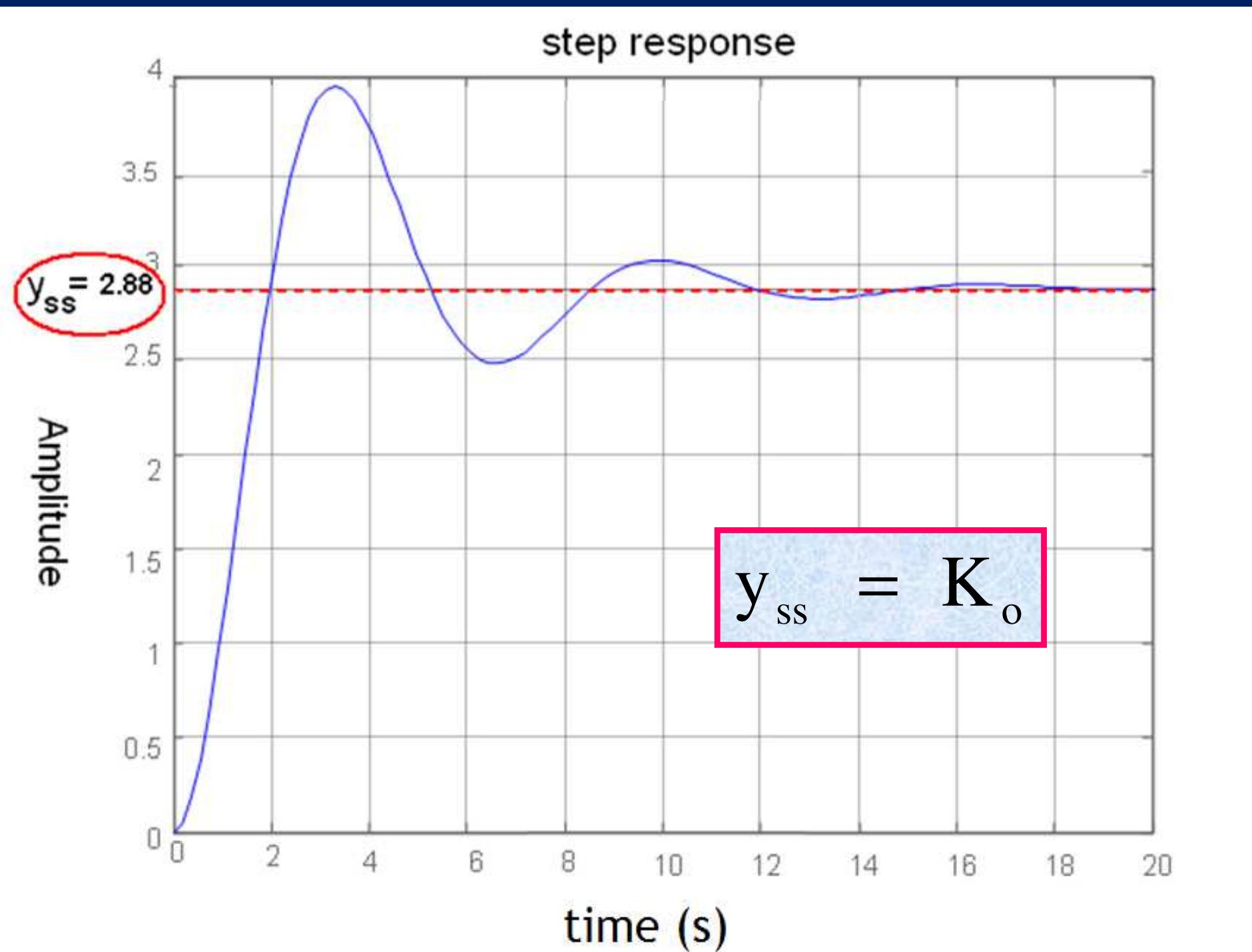
$$y_{ss} = \lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} s \cdot Y(s) =$$

$$= \lim_{s \rightarrow 0} \frac{\cdot K_o \omega_n^2 s}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot \frac{1}{s} =$$

$$= K_o$$

$y_{ss} = K_o$

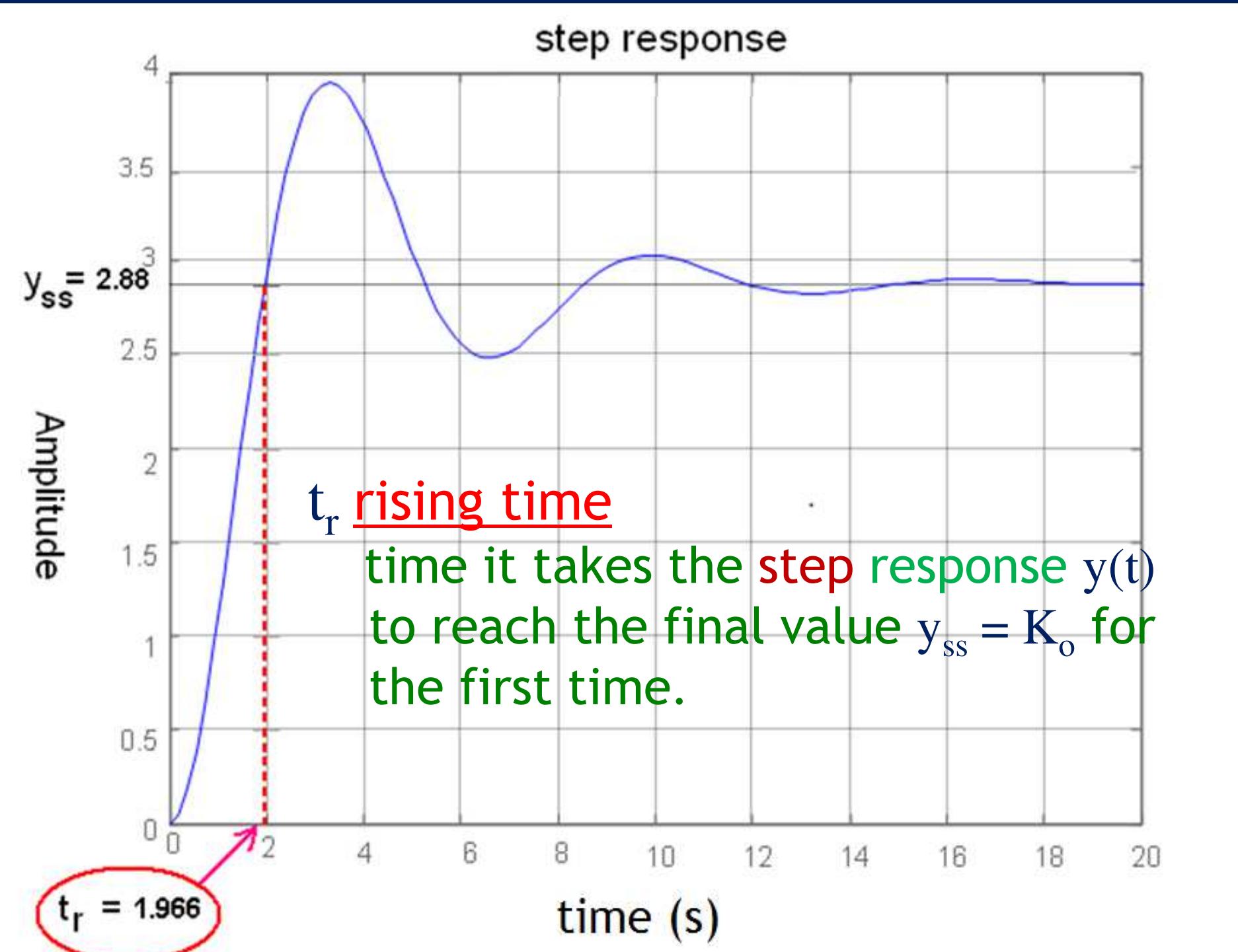
## Time domain analysis - 2<sup>nd</sup> order systems



*rising time*

$t_r$

## Time domain analysis - 2<sup>nd</sup> order systems



$t_r$  = rising time

Is the instant of time that  $y(t)$  reaches the final value  $K_o$  for the first time.

$$y(t_r) = K_o \left[ 1 - e^{-\zeta \omega_n t_r} \left( \cos \omega_d t_r + \frac{\zeta}{\sqrt{1-\zeta^2}} \cdot \sin \omega_d t_r \right) \right] = K_o$$

$$e^{-\zeta \omega_n t_r} \left( \cos \omega_d t_r + \frac{\zeta}{\sqrt{1-\zeta^2}} \cdot \sin \omega_d t_r \right) = 0$$

$$\operatorname{tg}(\omega_d t_r) = \frac{-\sqrt{1-\zeta^2}}{\zeta} = \frac{-\omega_d}{\zeta \omega_n}$$

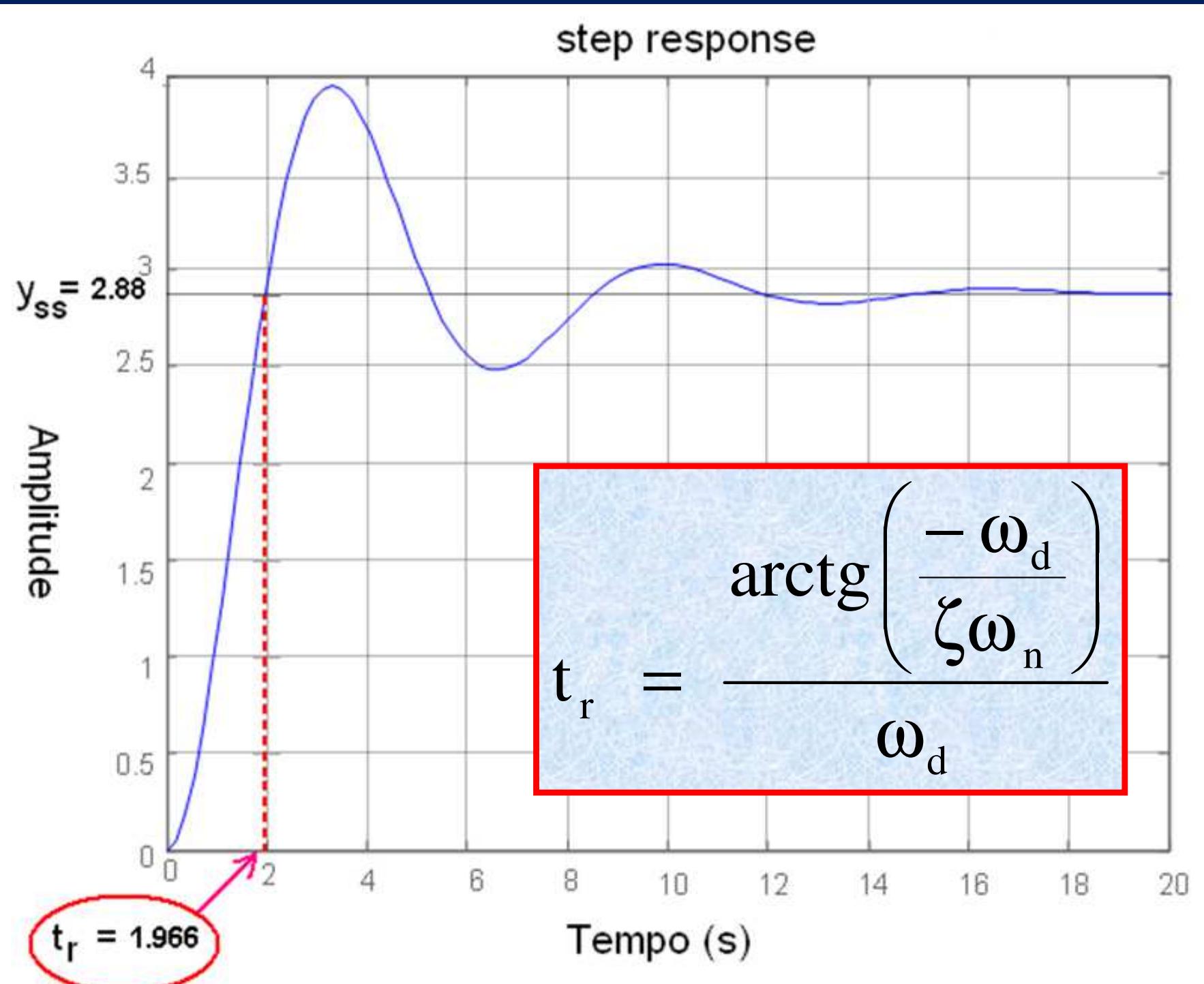
$t_r$  = rising time

Depends on the values of  $\zeta$  (*damping coefficient*),  
and of  $\omega_n$  (*natural frequency*)

$$t_r = \frac{\operatorname{arctg}\left(\frac{-\omega_d}{\zeta\omega_n}\right)}{\omega_d} = \frac{\operatorname{arctg}\left(\frac{-\sqrt{1-\zeta^2}}{\zeta}\right)}{\omega_d}$$

$$t_r = \operatorname{arctg}(-\omega_d/\zeta\omega_n) / \omega_d$$

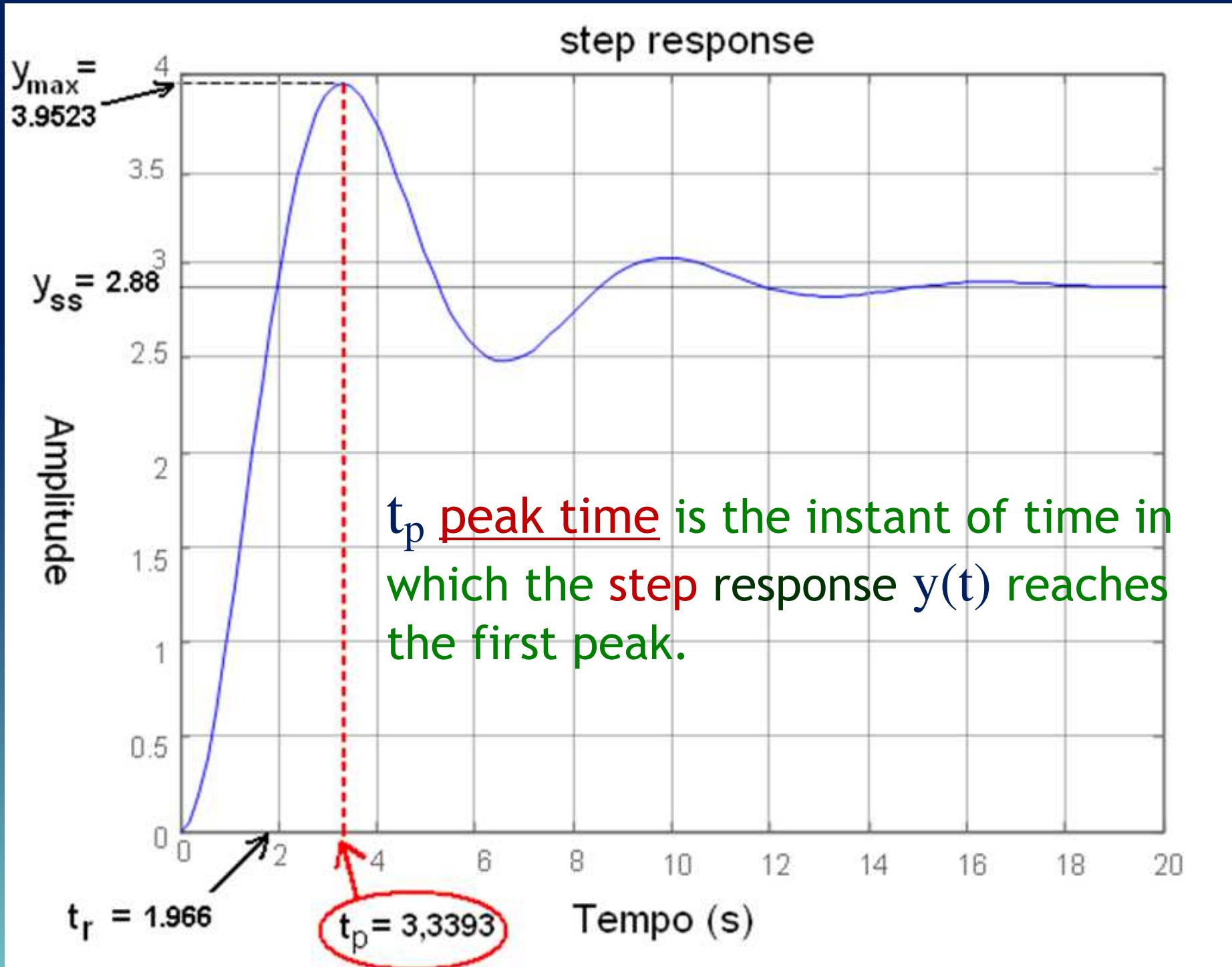
## Time domain analysis - 2<sup>nd</sup> order systems



*peak time*

$t_p$

## Time domain analysis - 2<sup>nd</sup> order systems



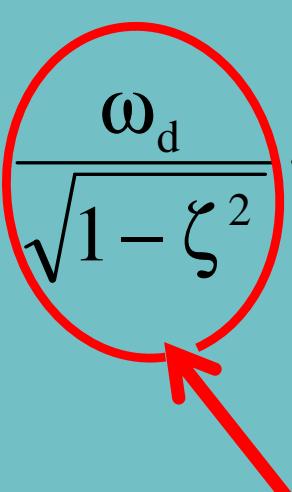
*peak time*

$y_{ss} = K_o$  (*gain*)

$\zeta$  (*damping coefficient*),

$\omega_n$  (*natural frequency*)

$$y' = \frac{dy}{dt} = K_o \left[ \zeta \omega_n \cdot e^{-\zeta \omega_n t} \left( \cos \omega_d t_r + \frac{\zeta}{\sqrt{1-\zeta^2}} \cdot \sin \omega_d t_r \right) + \right.$$

$$\left. - e^{-\zeta \omega_n t} \left( -\omega_d \sin \omega_d t_r + \zeta \frac{\omega_d}{\sqrt{1-\zeta^2}} \cdot \cos \omega_d t_r \right) \right]$$


$\omega_n$

*peak time*

$$\begin{aligned}
 \frac{dy}{dt} &= K_o \cdot e^{-\zeta \omega_n t} \left( \zeta \omega_n \cos \omega_d t + \frac{\zeta^2 \omega_n}{\sqrt{1-\zeta^2}} \cdot \sin \omega_d t + \right. \\
 &\quad \left. + \omega_d \sin \omega_d t - \zeta \omega_n \cdot \cos \omega_d t \right) \\
 &= K_o \cdot e^{-\zeta \omega_n t} \cdot \sin \omega_d t \cdot \left( \frac{\zeta^2 \omega_n}{\sqrt{1-\zeta^2}} + \frac{\omega_n (1-\zeta^2)}{\sqrt{1-\zeta^2}} \right) \\
 &= \cancel{K_o} \cdot \cancel{e^{-\zeta \omega_n t}} \cdot \sin \omega_d t \cdot \cancel{\frac{\omega_n}{\sqrt{1-\zeta^2}}} = 0
 \end{aligned}$$

*peak time*

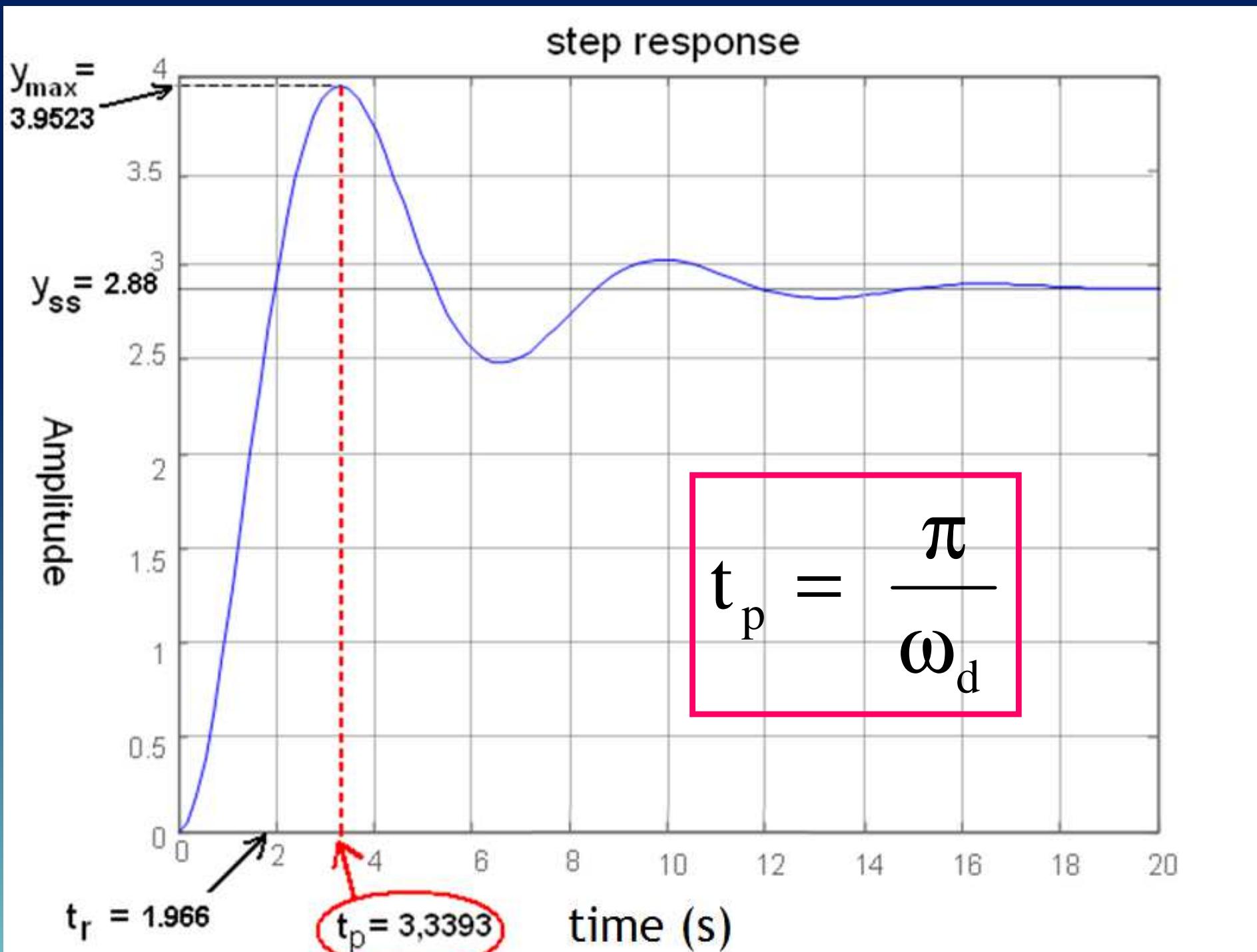
$$\frac{dy}{dt} = 0 \quad \longrightarrow \quad \sin \omega_d t = 0$$

$$\longrightarrow \omega_d t = 0, \pi, 2\pi, 3\pi, \dots$$

$$\longrightarrow t_p = \frac{\pi}{\omega_d}$$

$$t_p = \pi / \omega_d$$

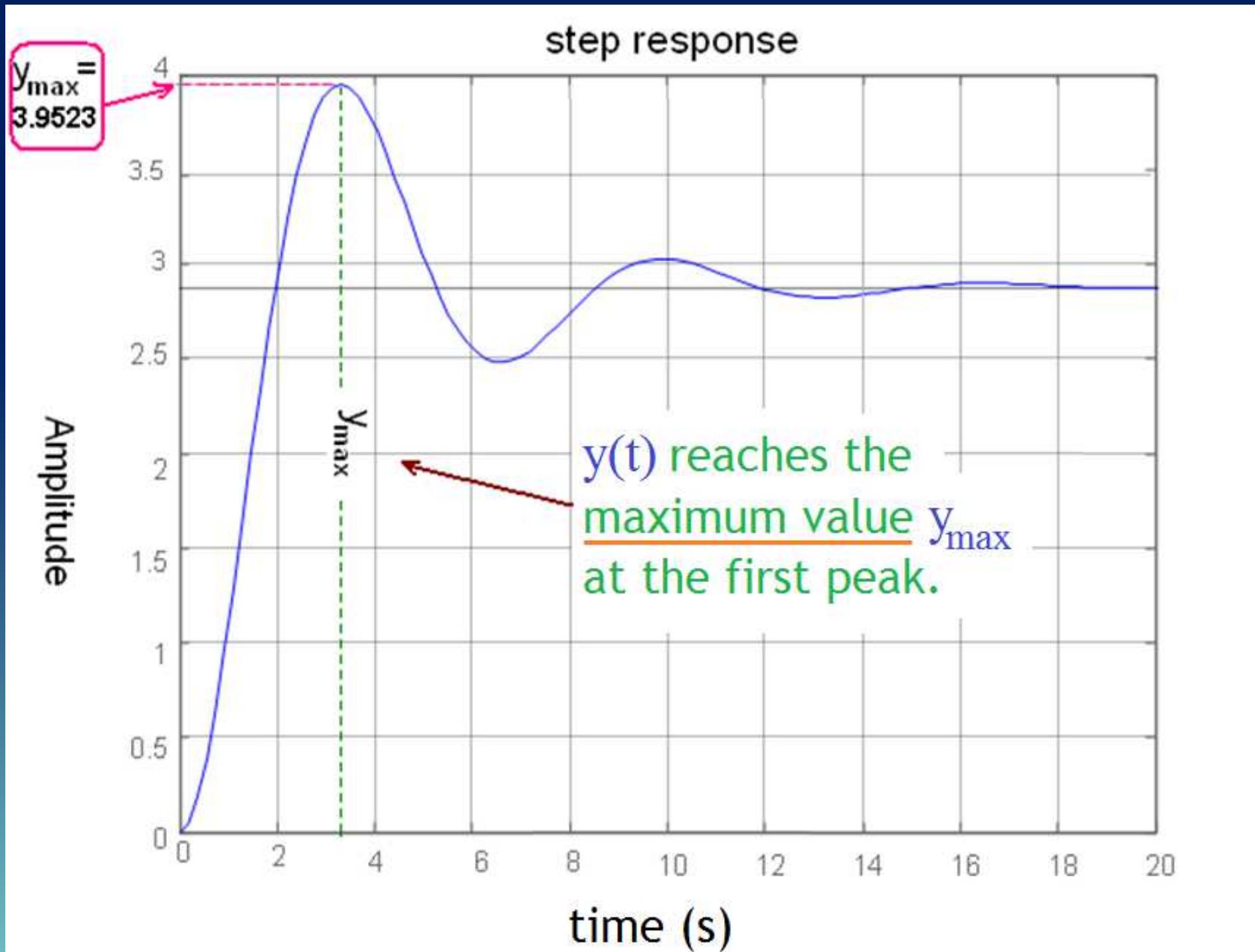
## Time domain analysis - 2<sup>nd</sup> order systems



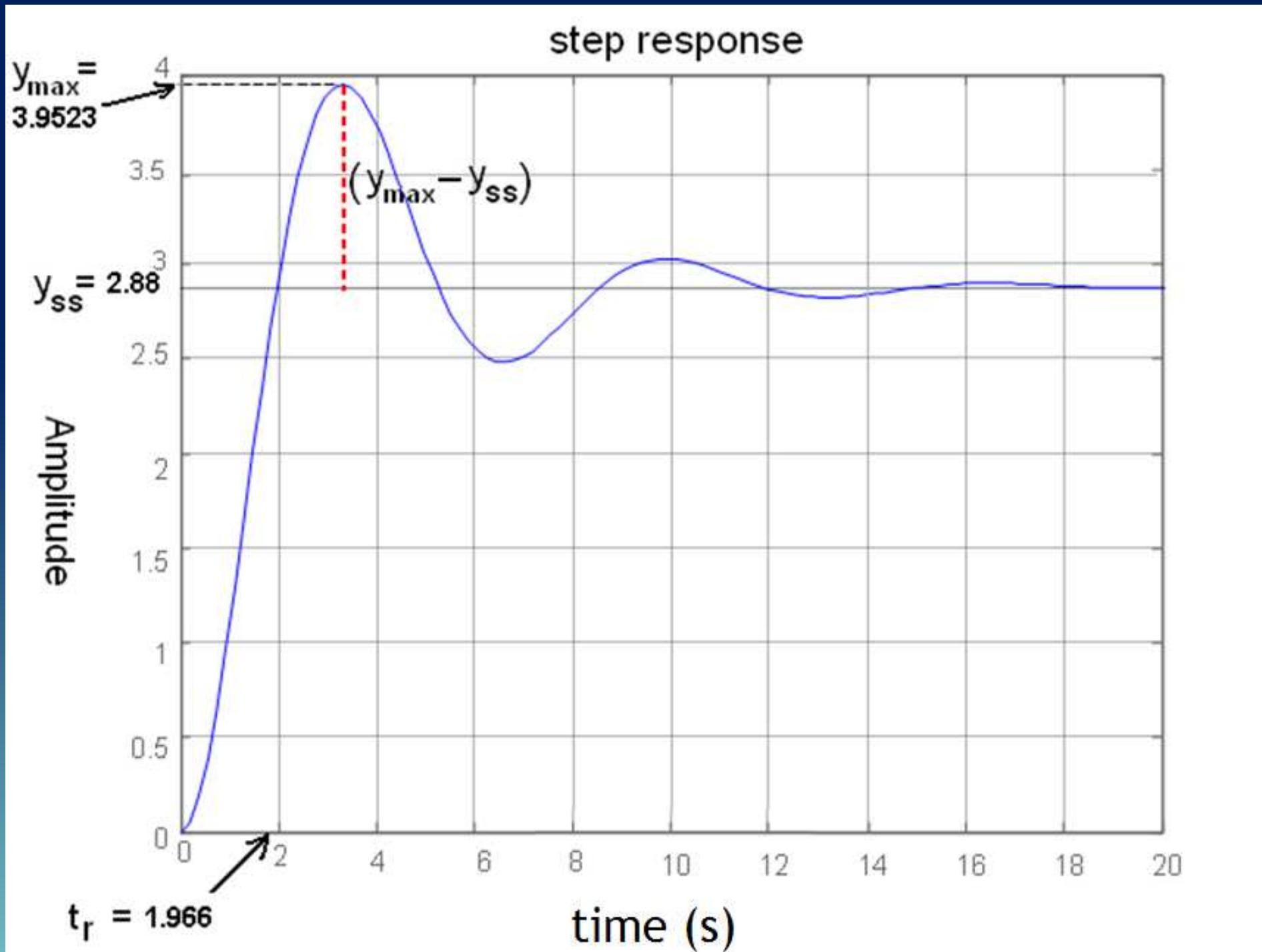
overshoot

$M_p$

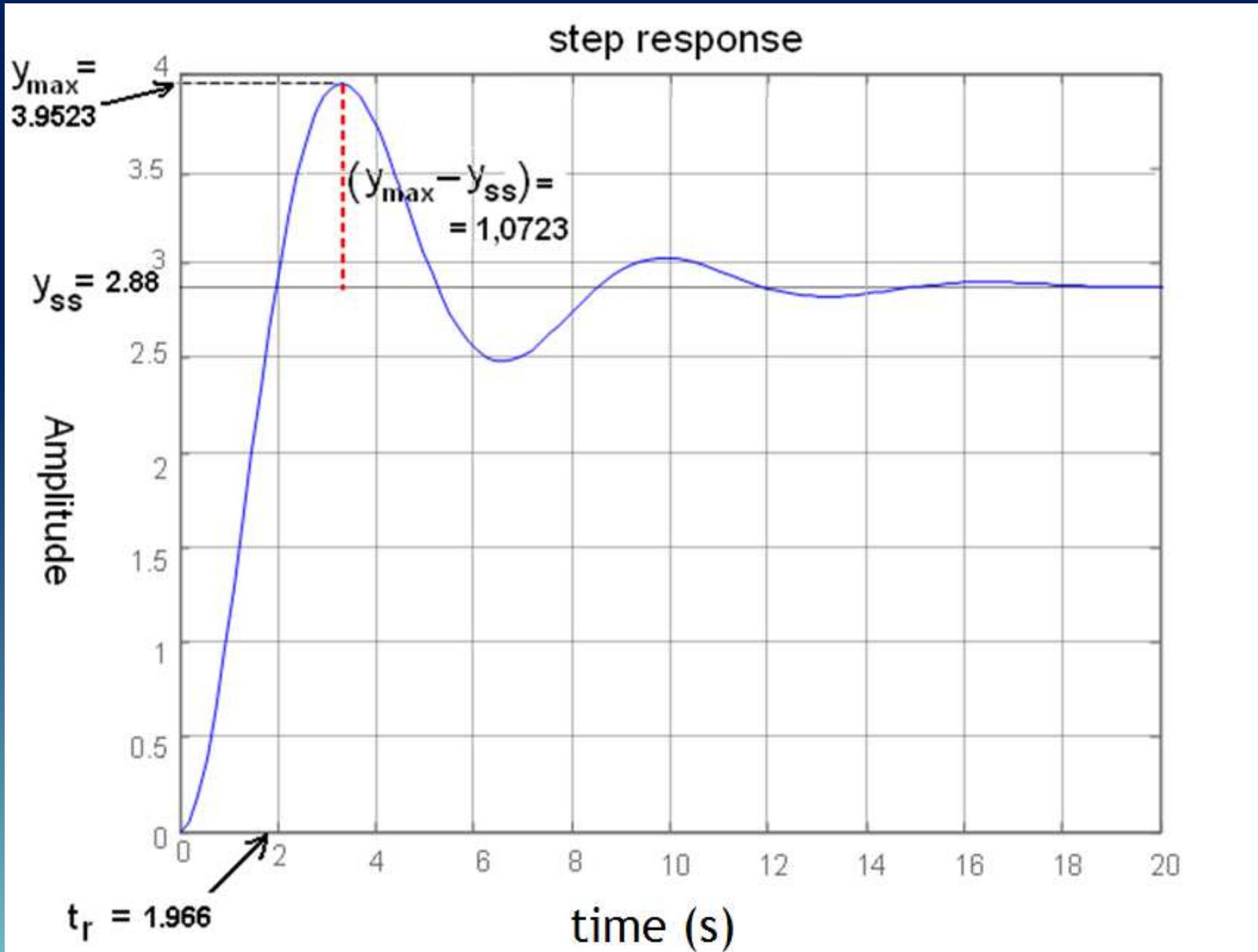
## Time domain analysis - 2<sup>nd</sup> order systems



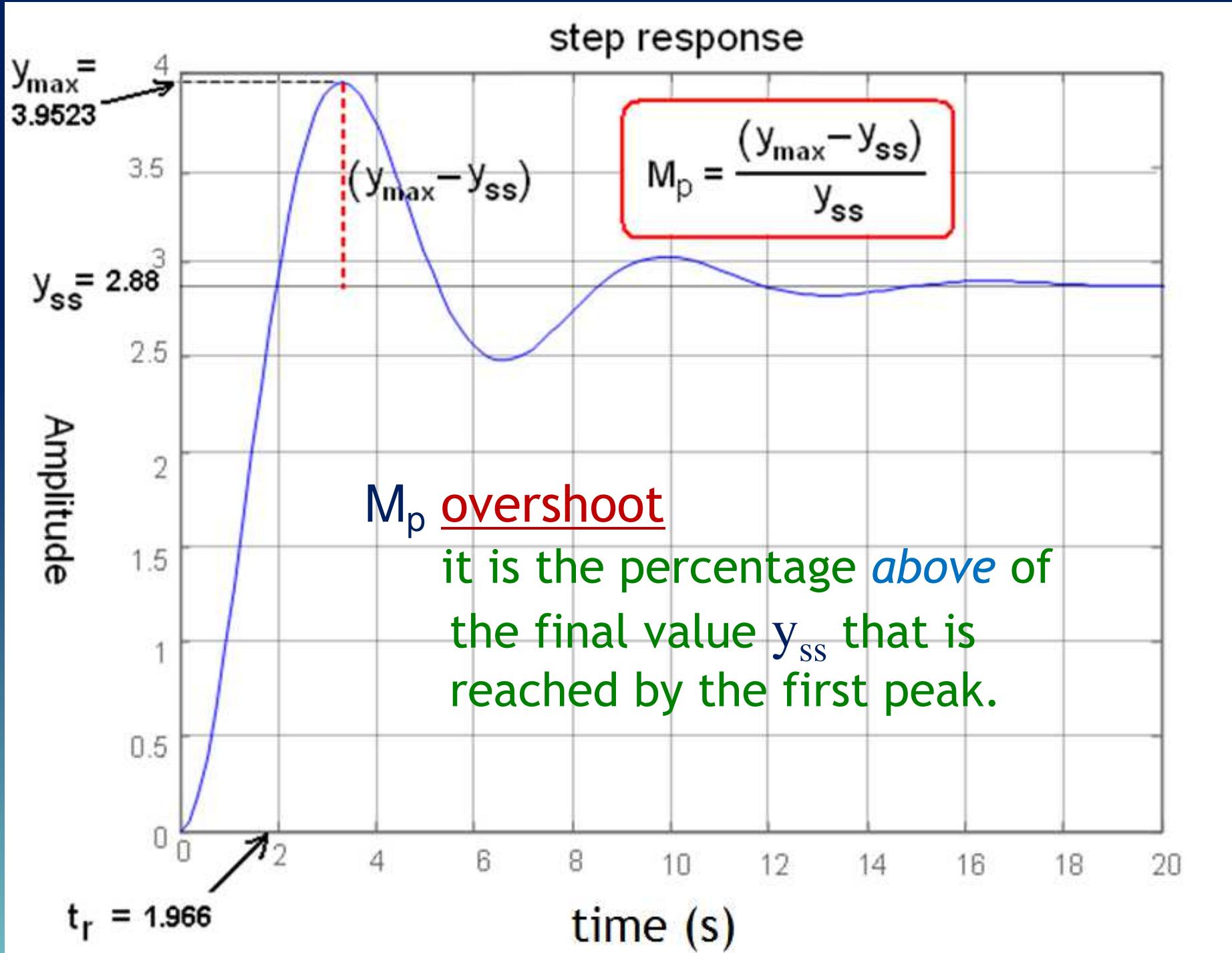
## Time domain analysis - 2<sup>nd</sup> order systems



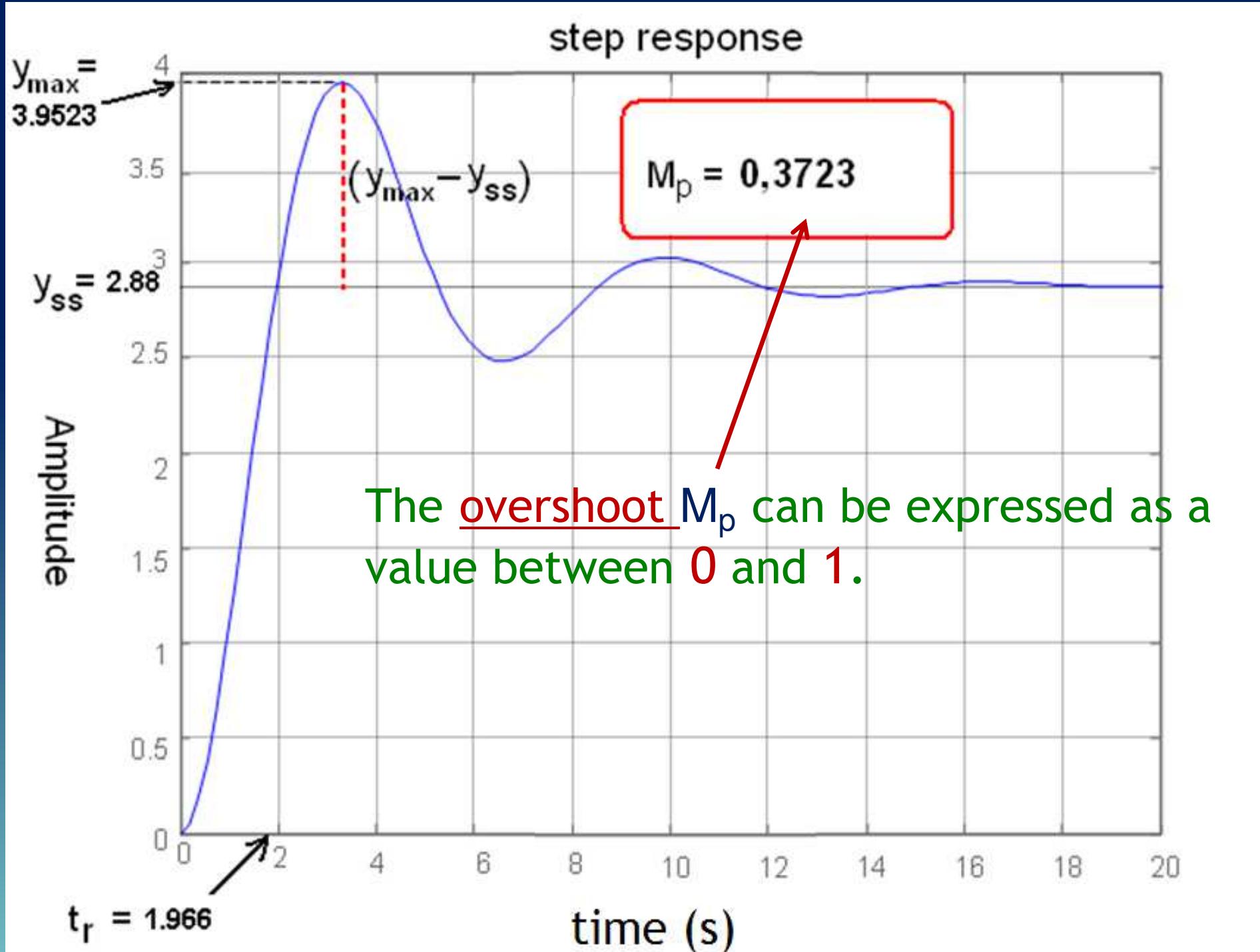
## Time domain analysis - 2<sup>nd</sup> order systems



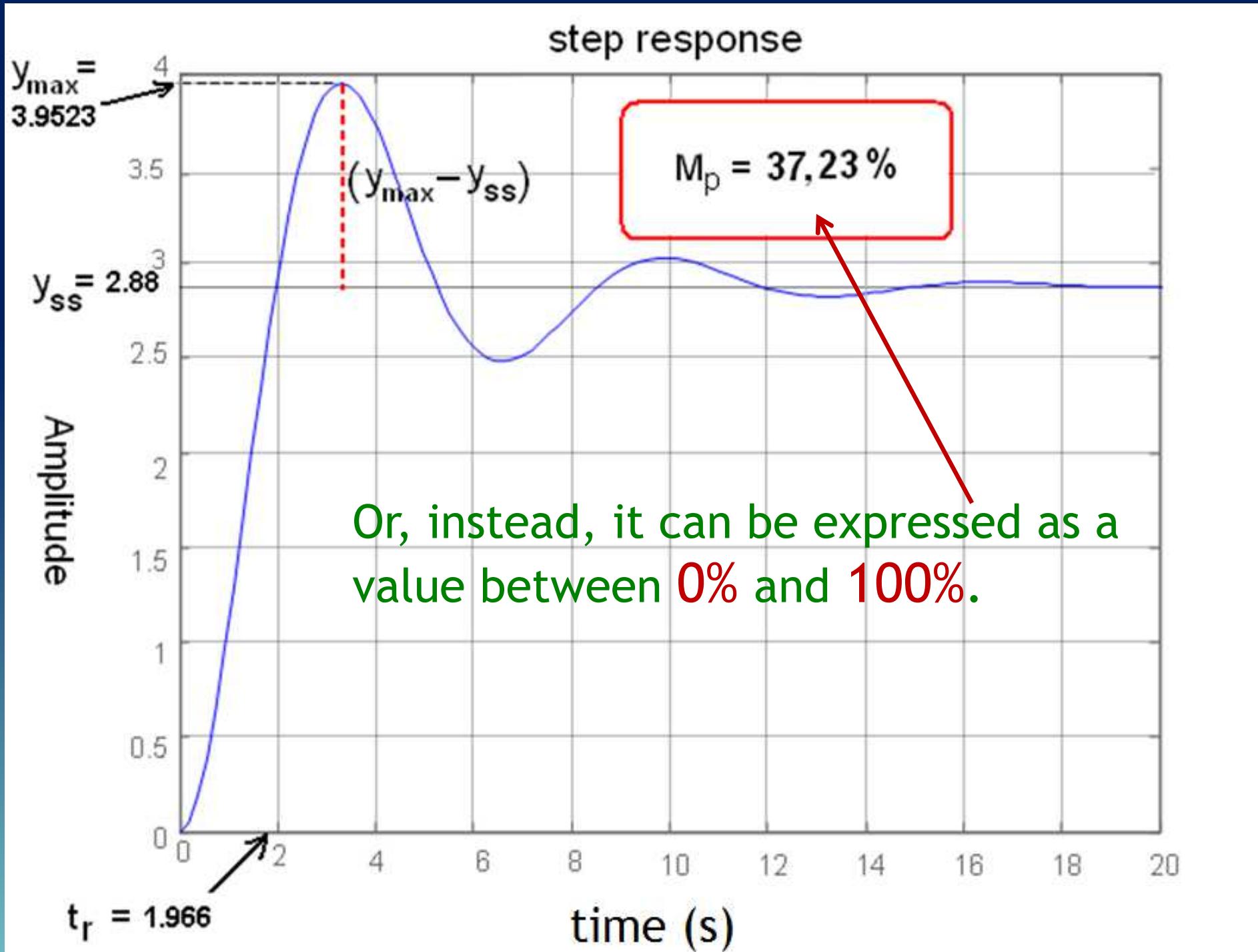
## Time domain analysis - 2<sup>nd</sup> order systems



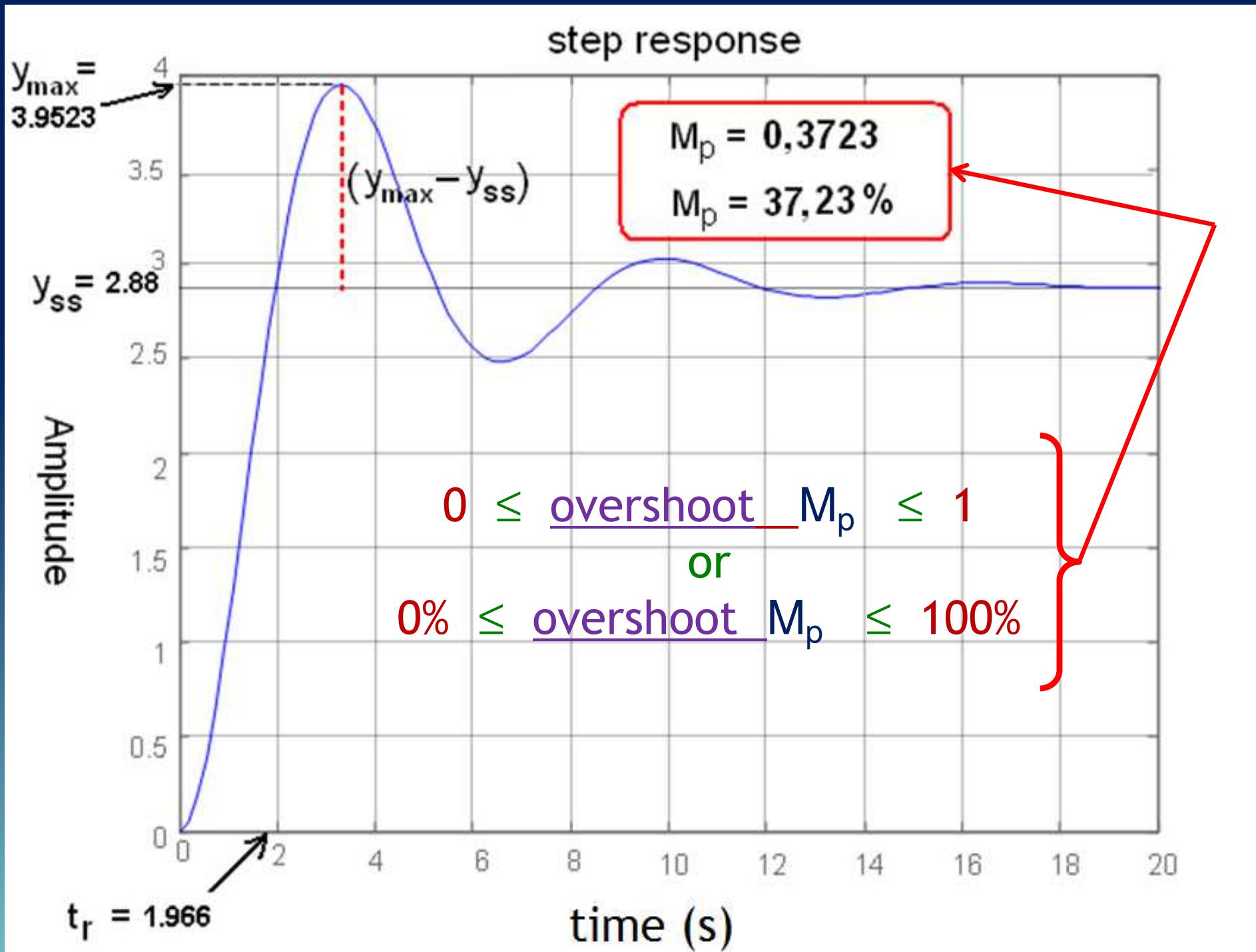
## Time domain analysis - 2<sup>nd</sup> order systems



## Time domain analysis - 2<sup>nd</sup> order systems



## Time domain analysis - 2<sup>nd</sup> order systems



overshoot

$y_{ss} = K_o$  (*gain*)

$\zeta$  (*damping coefficient*),

$\omega_n$  (*natural frequency*)

$$M_p = \frac{y_{max} - y_{ss}}{y_{ss}} \quad \text{ou} \quad M_p = \frac{y(t_p) - K_o}{K_o}$$

$$M_p = \frac{K_o \left[ 1 - e^{-\zeta \omega_n t_p} \left( \underbrace{\cos(\pi)}_{-1} - \frac{\cancel{\sin(\pi)}}{\sqrt{1-\zeta^2}} \right) \right] - K_o}{K_o}$$

~~$$M_p = \frac{K_o + K_o e^{-\zeta \omega_n (\pi/\omega_d)} - K_o}{K_o}$$~~

overshoot

$$M_p = \frac{K_o e^{\frac{-\zeta \pi}{\sqrt{1 - \zeta^2}}}}{K_o}$$

→  $M_p = e^{\frac{-\zeta \pi}{\sqrt{1 - \zeta^2}}}$

$M_p$  depends only on  $\zeta$  (*damping coefficient*)

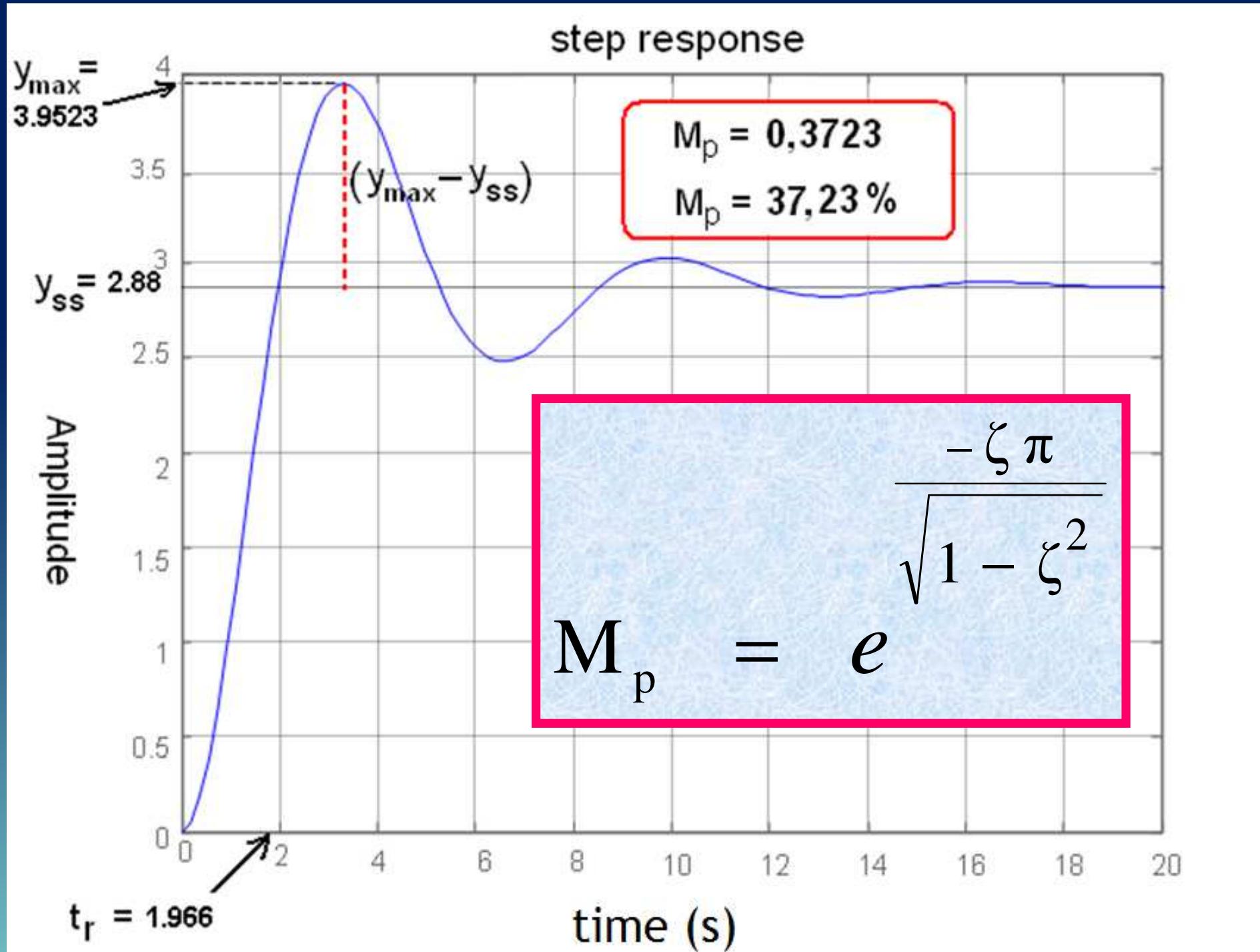
overshoot

$M_p$  depends only on  $\zeta$  (*damping coefficient*)

$$M_p = e^{\frac{-\zeta \pi}{\sqrt{1 - \zeta^2}}} \quad \text{or}$$

$$M_p = e^{\frac{-\zeta \pi}{\sqrt{1 - \zeta^2}}} \times 100\%$$

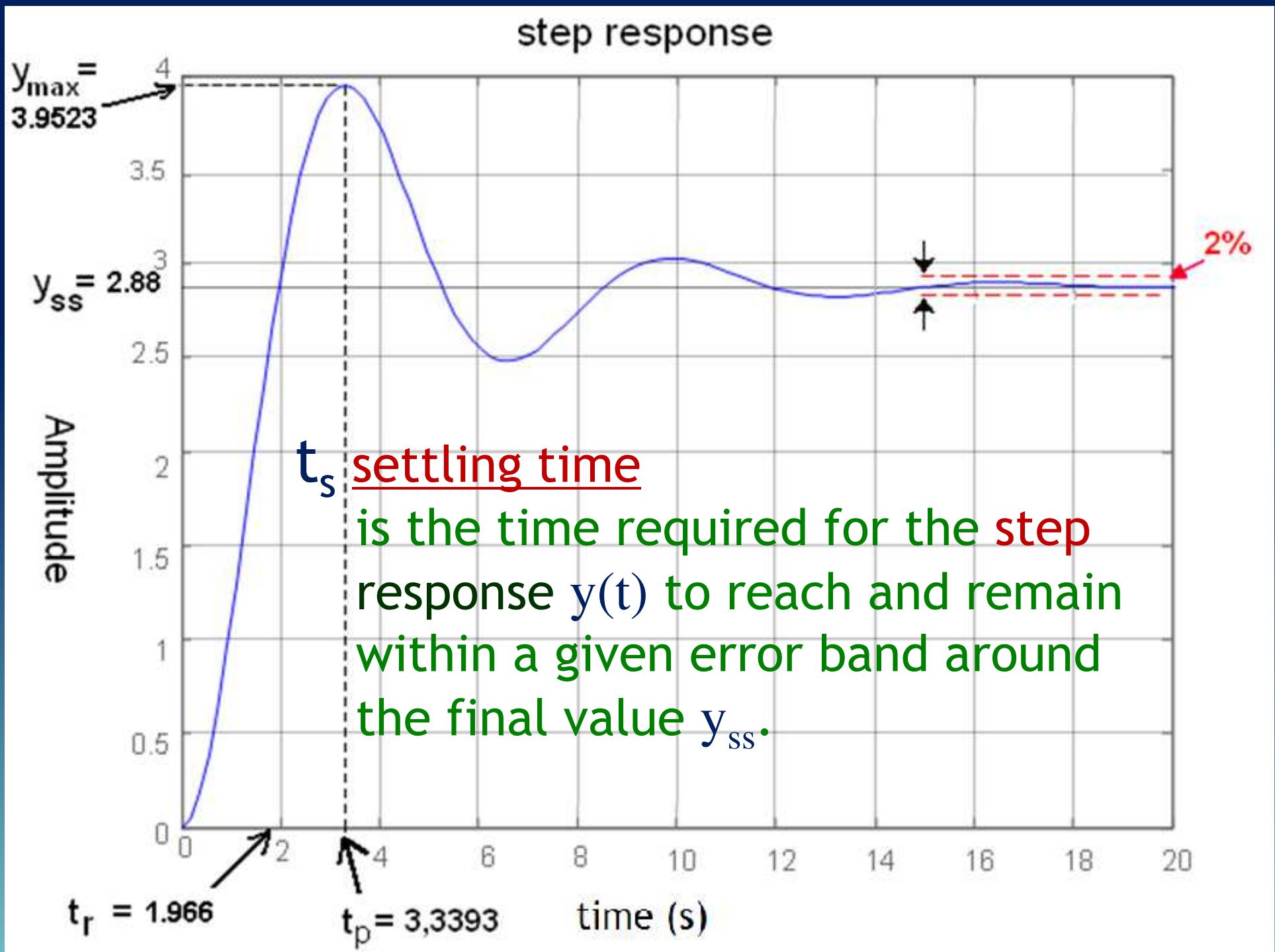
## Time domain analysis - 2<sup>nd</sup> order systems



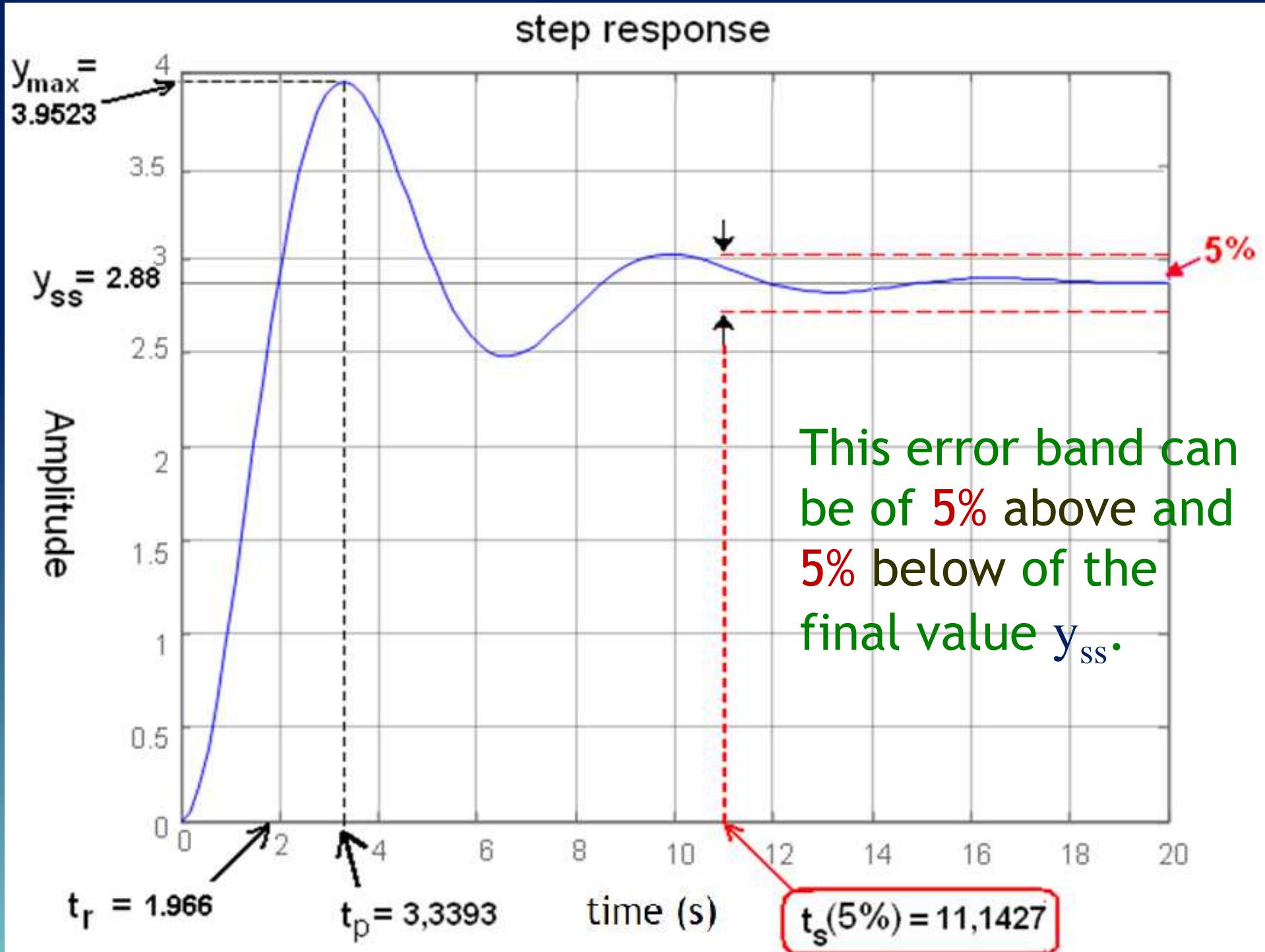
settling time

$t_s$

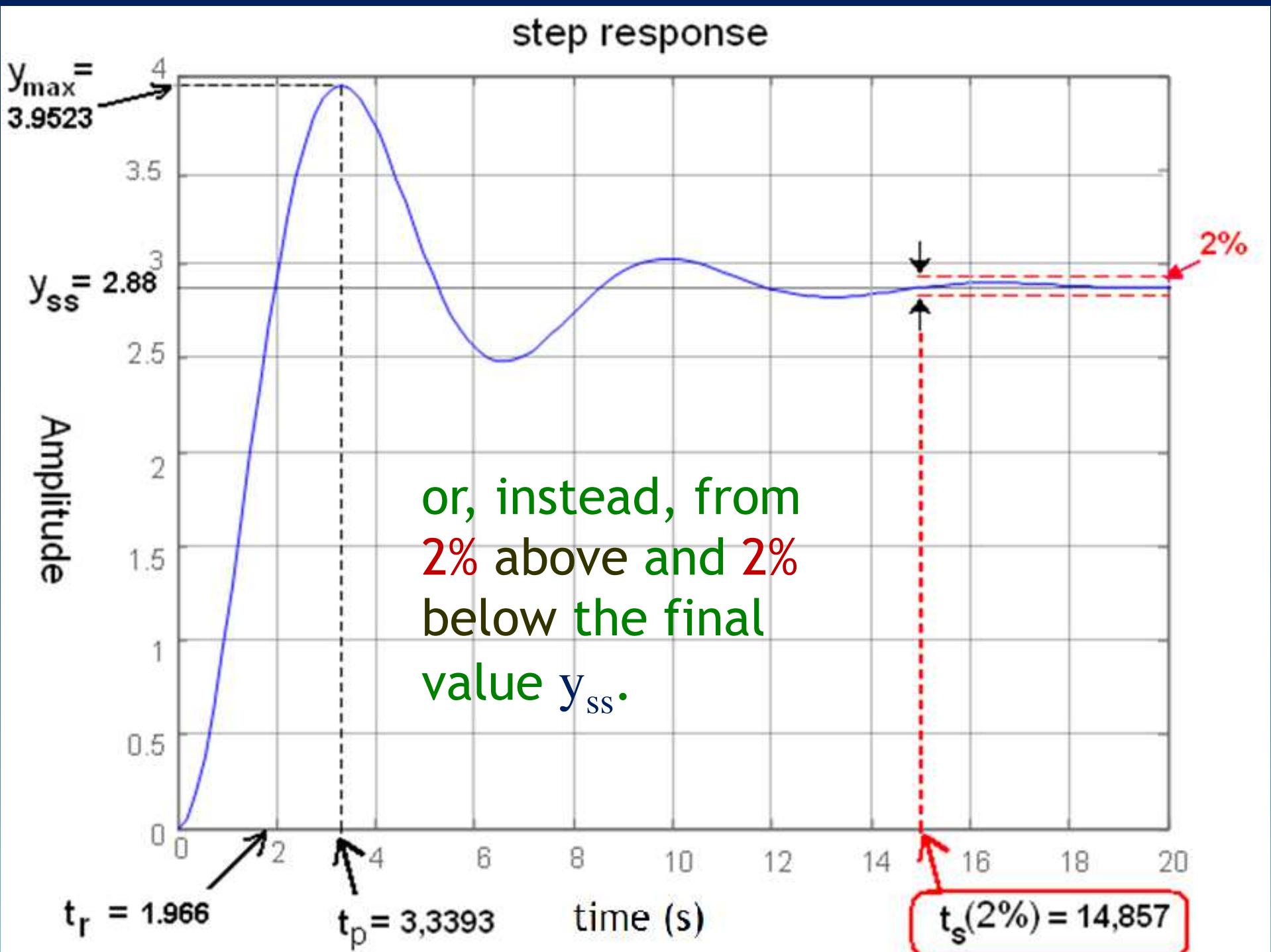
## Time domain analysis - 2<sup>nd</sup> order systems



## Time domain analysis - 2<sup>nd</sup> order systems

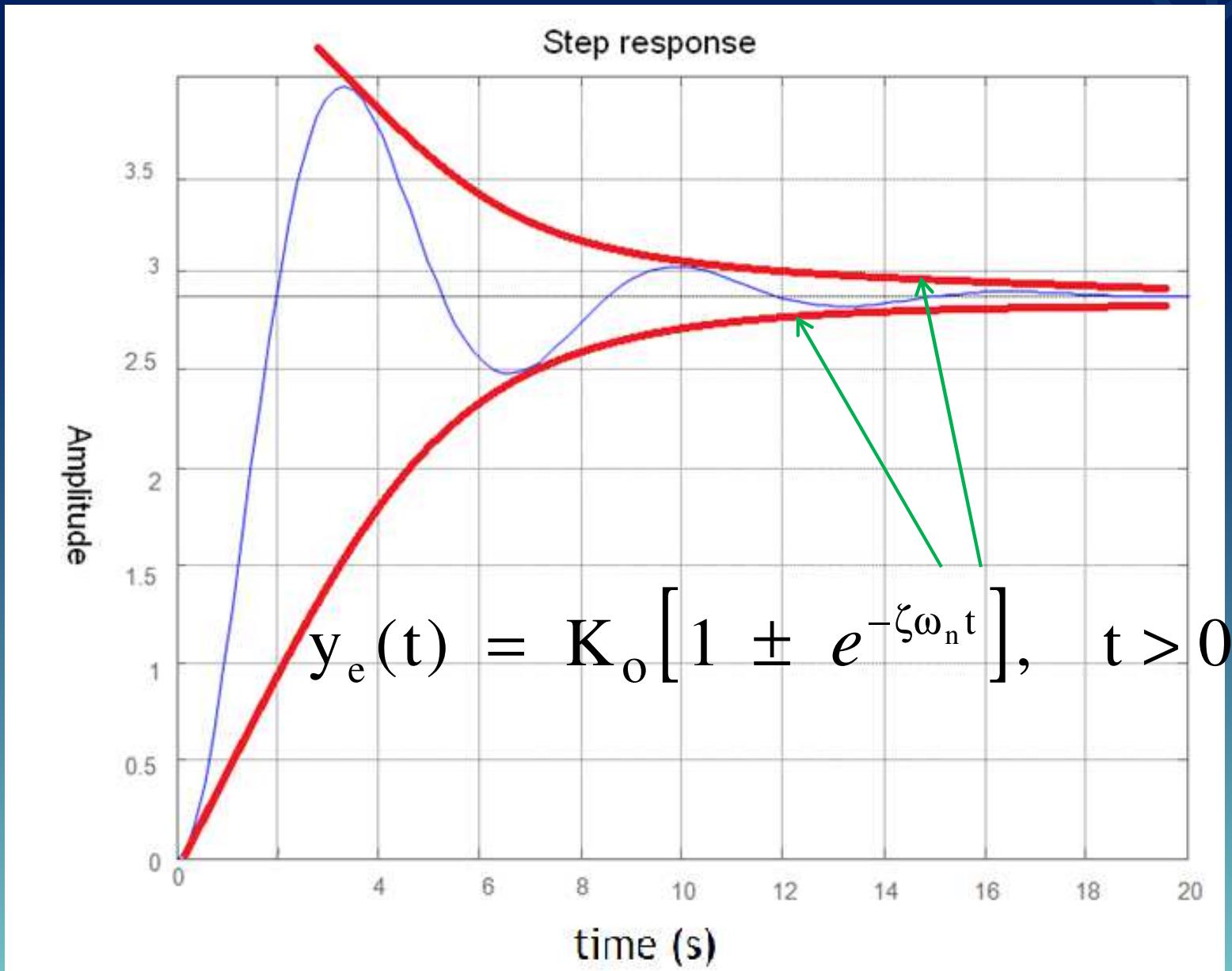


## Time domain analysis - 2<sup>nd</sup> order systems



## Time domain analysis - 2<sup>nd</sup> order systems

The settling time is obtained from the equations of  $y_e(t)$ , the curves that encompass  $y(t)$ .



That is, the *settling time*  $t_s$  is obtained by calculating

$$y_e(t_s) \approx 1,05 K_o$$

for the case of  $t_s$  with 5% tolerance, and

$$y_e(t_s) \approx 1,02 K_o$$

for the case of  $t_s$  with 2% tolerance,

obtaining the following values:

$$t_s(5\%) = \frac{3}{\zeta \omega_n}$$

$$t_s(2\%) = \frac{4}{\zeta \omega_n}$$

## settling time

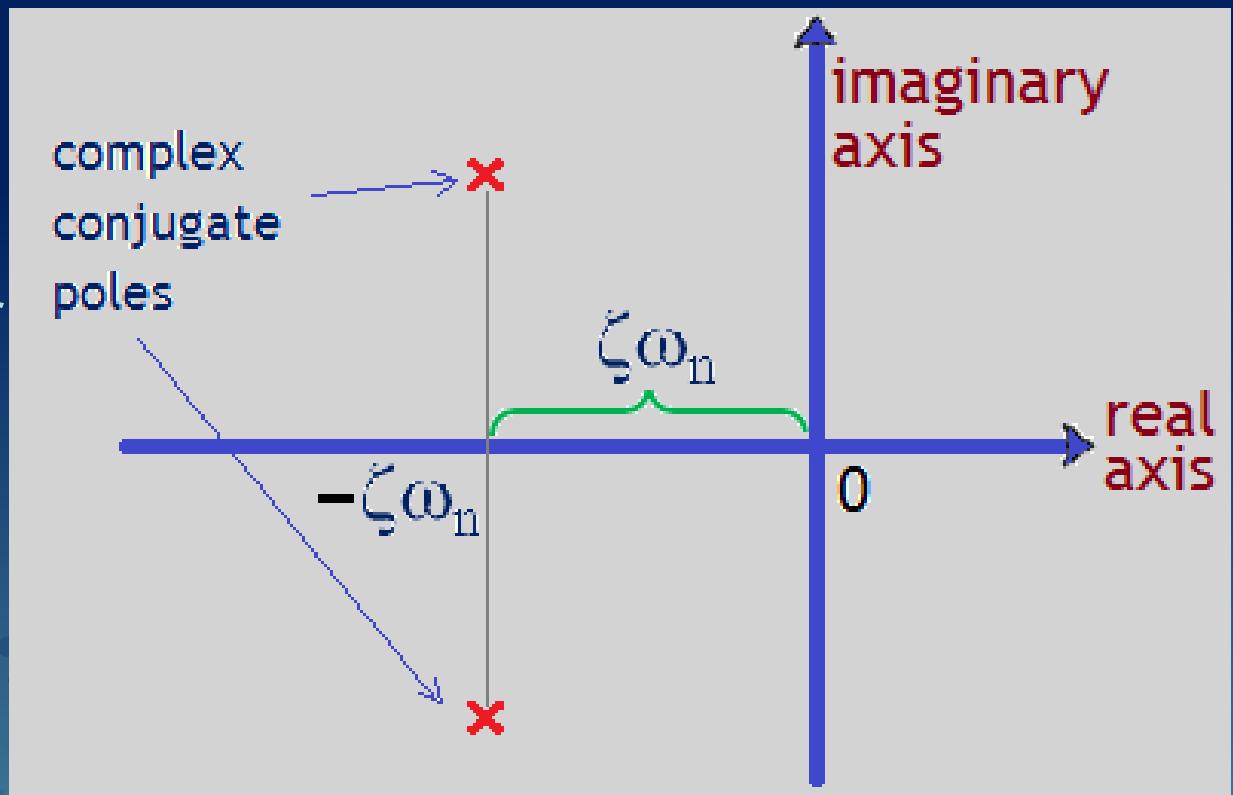
Note that the settling time  $t_r$  is inversely proportional to  $\zeta\omega_n$ , which is the distance of the real part of the poles to the origin.

hence:

$$t_s(5\%) = \frac{3}{\zeta\omega_n}$$

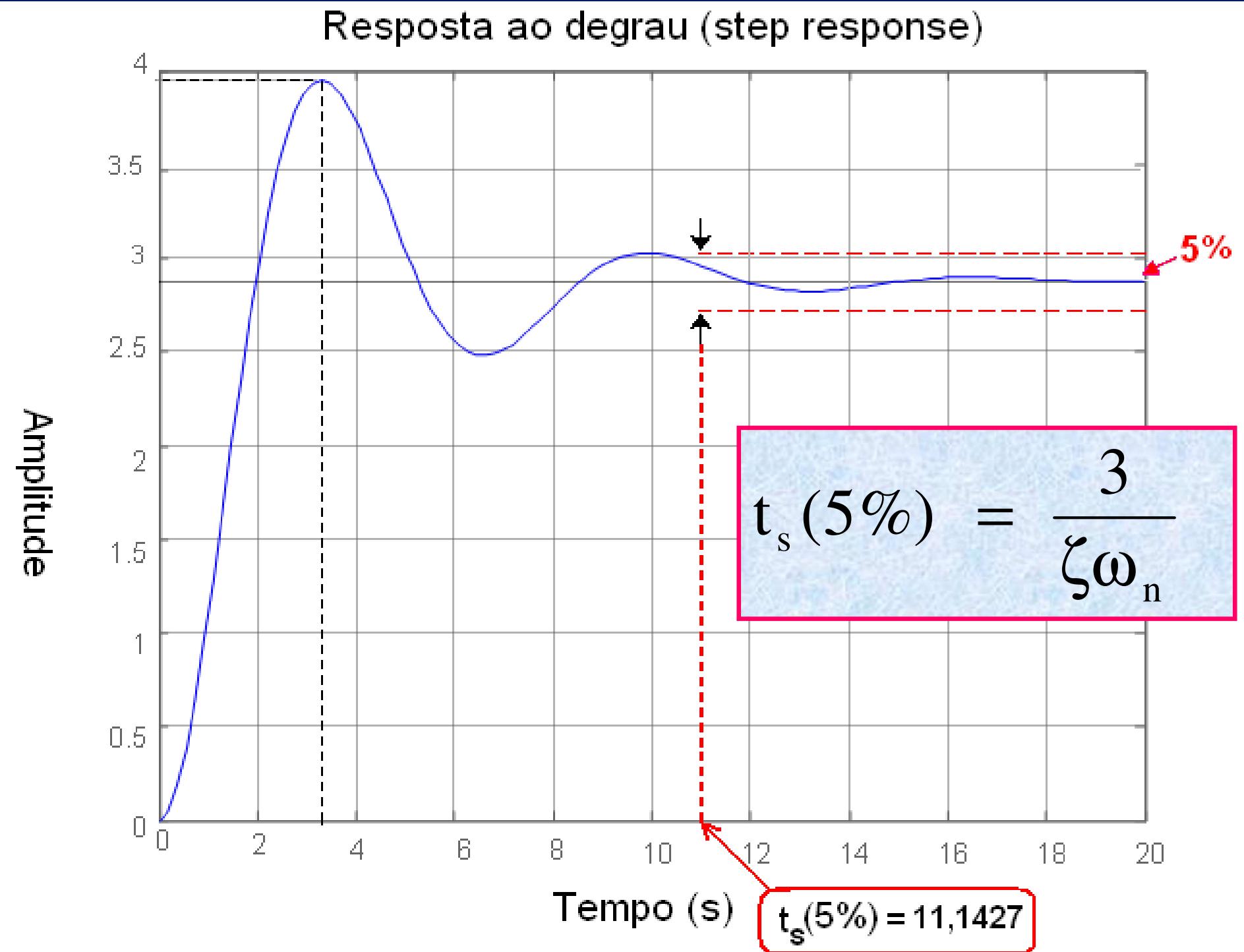
and

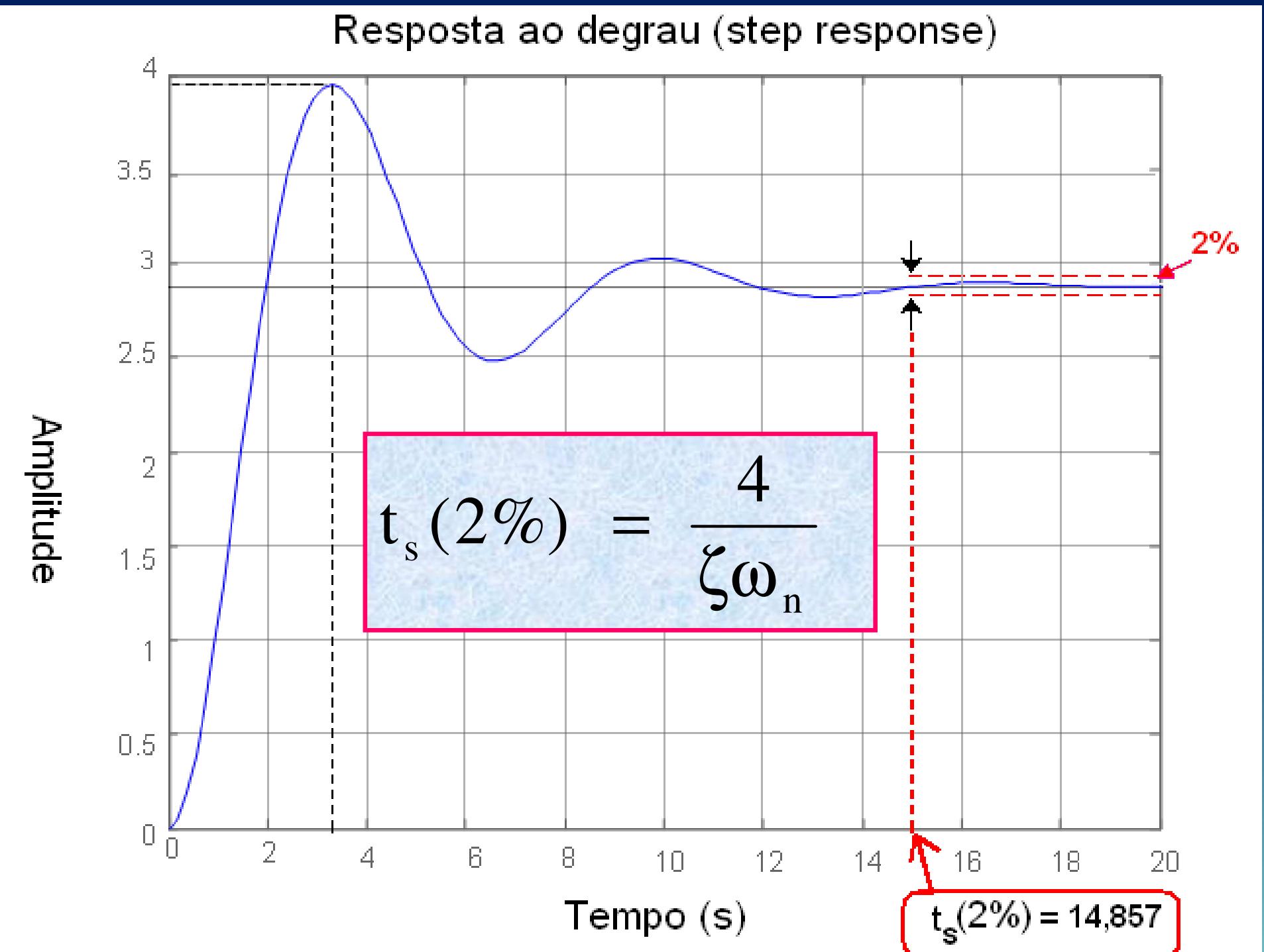
$$t_s(2\%) = \frac{4}{\zeta\omega_n}$$



$$t_s(5\%) = 3 / \zeta\omega_n$$

$$t_s(2\%) = 4 / \zeta\omega_n$$





Note that we have seen cases in which

$$0 < \zeta < 1$$

$$\zeta = 1$$

$$\zeta > 1$$

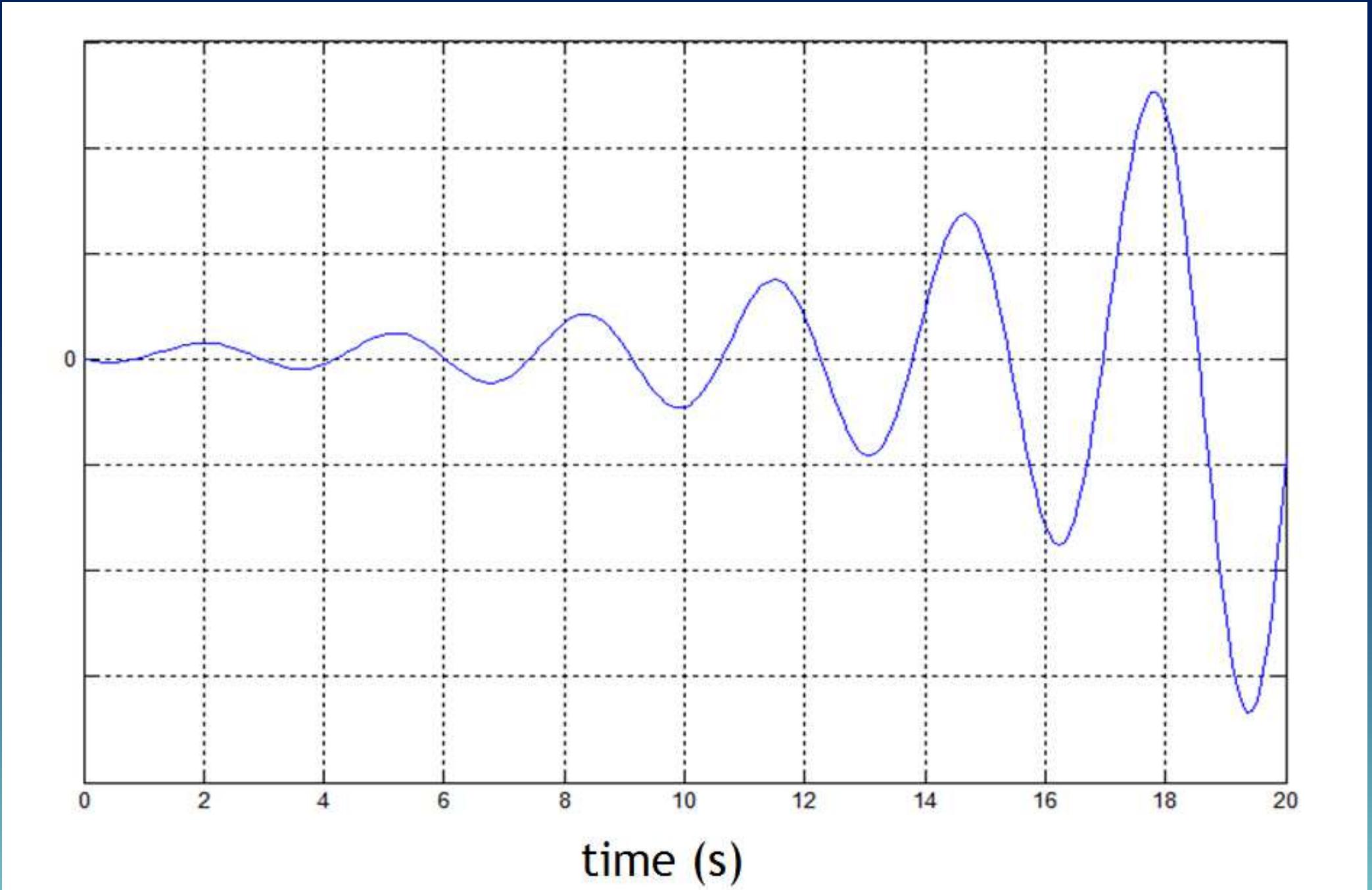
that is:

$$\zeta > 0$$

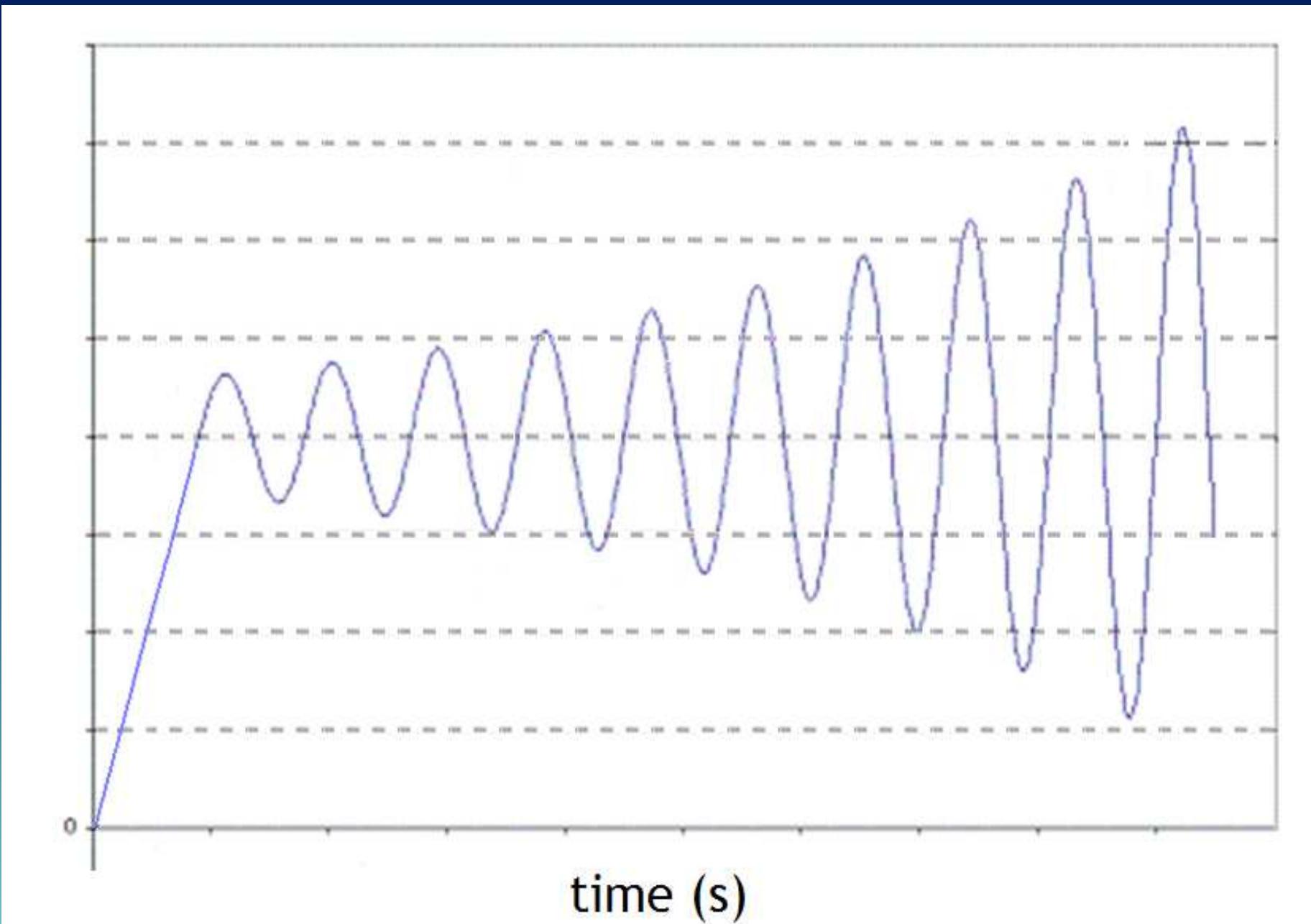
However, if  $\zeta < 0$  then:

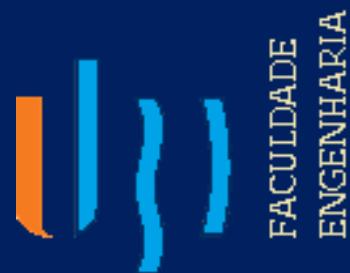
**the system is unstable**

$\zeta < 0 \rightarrow$  unstable system (an example)



$\zeta < 0 \rightarrow$  **unstable system** (another example)





FACULDADE  
ENGENHARIA

Departamento de  
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Thank you!

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